IEC 60034-14

Edition 4.0 2018-08
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INTERNATIONAL STANDARD

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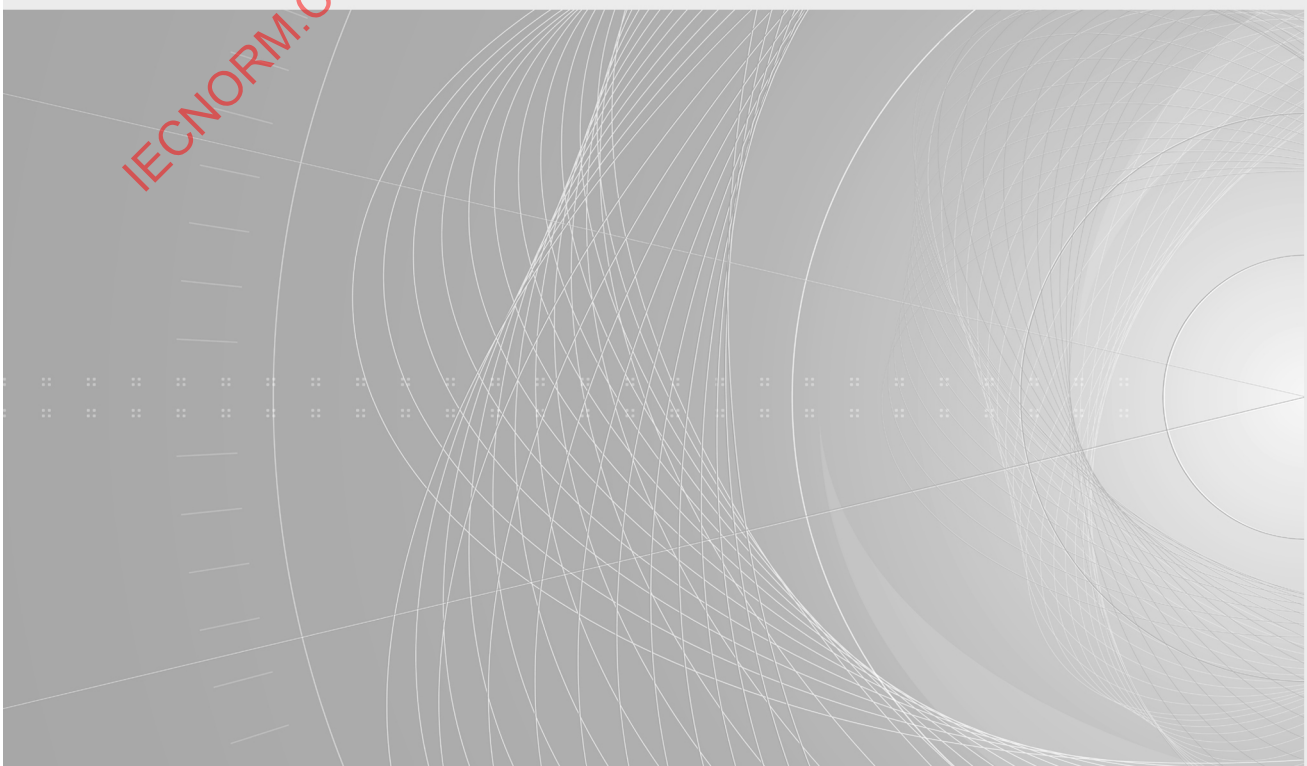
Rotating electrical machines –
Part 14: Mechanical vibration of certain machines with shaft heights 56 mm and higher – Measurement, evaluation and limits of vibration severity



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INTERNATIONAL ELECTROTECHNICAL COMMISSION

ICS 29.160.01

ISBN 978-2-8322-5995-5

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INTERNATIONAL ELECTROTECHNICAL COMMISSION

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ROTATING ELECTRICAL MACHINES –

Part 14: Mechanical vibration of certain machines with shaft heights 56 mm and higher – Measurement, evaluation and limits of vibration severity

FOREWORD

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This redline version of the official IEC Standard allows the user to identify the changes made to the previous edition. A vertical bar appears in the margin wherever a change has been made. Additions are in green text, deletions are in strikethrough red text.

International Standard IEC 60034-14 has been prepared by IEC technical committee 2: Rotating machinery.

This fourth edition cancels and replaces the third edition, published in 2003, and its amendment 1, published in 2007. It constitutes a technical revision.

The significant technical changes with respect to the previous edition are:

a) 6.2 is significantly changed to better explain the definition “free suspension”.
b) 6.3: a second method of rigid mount is added since the first method is not always possible on the test floor.
c) 7.1: an improved option for shaft key is defined.
d) Clause 8: considerable effort to harmonize with NEMA MG 1 and IEEE 841 and API 541, and also establish levels which are achievable and more in line with best practices. Table 1 is reduced to two shaft-height range sections.
e) 8.2: definition of twice line frequency simplified along with Figure 7 added.

The text of this International Standard is based on the following documents:

FDIS	Report on voting
2/1906/FDIS	2/1914/RVD

Full information on the voting for the approval of this International Standard can be found in the report on voting indicated in the above table.

This document has been drafted in accordance with the ISO/IEC Directives, Part 2.

A list of all parts in the IEC 60034 series, published under the general title *Rotating electrical machines*, can be found on the IEC website.

NOTE For A table of cross-references of all IEC TC 2 publications can be found in the IEC TC 2 dashboard on the IEC website.

The committee has decided that the contents of this document will remain unchanged until the stability date indicated on the IEC website under "http://webstore.iec.ch" in the data related to the specific document. At this date, the document will be

- reconfirmed,
- withdrawn,
- replaced by a revised edition, or
- amended.

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ROTATING ELECTRICAL MACHINES –

Part 14: Mechanical vibration of certain machines with shaft heights 56 mm and higher – Measurement, evaluation and limits of vibration severity

1 Scope

This part of IEC 60034 specifies the factory acceptance vibration test procedures and vibration limits for certain electrical machines under specified conditions, when uncoupled from any load or prime mover.

It is applicable to DC and three-phase AC machines, with shaft heights 56 mm and higher and a rated output up to 50 MW, at operational speeds from 120 min^{-1} up to and including 15 000 min^{-1}.

This document is not applicable to machines mounted *in situ* (on site), three-phase commutator motors, single-phase machines, three-phase machines operated on single-phase systems, vertical waterpower generators, turbine generators greater than 20 MW and machines with magnetic bearings or series-wound machines.

NOTE For machines measured *in situ,* refer to applicable parts of ISO 20816, ISO 10816 and ISO 7919.

2 Normative references

The following documents are referred to in the text in such a way that some or all of their content constitutes requirements of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

IEC 60034-1, *Rotating electrical machines – Part 1: Rating and performance*

IEC 60034-7, *Rotating electrical machines – Part 7: Classification of types ofconstructions and mounting arrangements ~~and terminal box position~~ (IM Code)*

ISO 2954, *Mechanical vibration of rotating and reciprocating machinery – Requirements for instruments for measuring vibration severity*

~~ISO 7919-1, *Mechanical vibration of non-reciprocating machines – Measurements on rotating shafts and evaluation criteria – Part 1: General guidelines*~~

~~ISO 8821, *Mechanical vibration – Balancing – Shaft and fitment key convention*~~

ISO 10817-1, *Rotating shaft vibration measuring systems – Part 1: Relative and absolute sensing of radial vibration from rotating shafts*

ISO 20816-1, *Mechanical vibration – Measurement and evaluation of machine vibration – Part 1: General guidelines*

ISO 21940-32, *Mechanical vibration – Rotor balancing – Part 32: Shaft and fitment key convention*

3 Terms and definitions

For the purposes of this document, the terms and definitions given in IEC 60034-1 apply.

ISO and IEC maintain terminological databases for use in standardization at the following addresses:

- IEC Electropedia: available at http://www.electropedia.org/
- ISO Online browsing platform: available at http://www.iso.org/obp

4 Measurement quantities

4.1 General

Measurement quantities are the vibration displacement~~, vibration acceleration~~ and vibration velocity at the machine bearings and the relative shaft vibration displacement within or near to the machine bearings.

4.2 Vibration magnitude

The criterion for the vibration magnitude at the machine bearings shall be the broadband r.m.s. value of the vibration displacement in micrometres or the vibration velocity in millimeters per second~~ and the vibration acceleration in meters per second squared~~ in the frequency range specified in Clause 5. The ~~greatest~~ maximum value, determined at the prescribed measurement positions and prescribed measuring variable, according to this standard, characterizes the vibration magnitude of the machine.

Induction motors (especially of two pole type) frequently show vibration beating at twice slip frequency. In these cases, the decisive vibration magnitude shall be determined from the relationship:

$$x_{\text{r.m.s.}} = \sqrt{\frac{x_{\text{max}}^{\ 2} + x_{\text{min}}^{\ 2}}{2}}$$

where

x_{max} is the maximum r.m.s. value of vibration displacement or vibration velocity~~ or acceleration~~;

x_{min} is the minimum r.m.s. value of vibration displacement or vibration velocity~~ or acceleration~~.

The sample rate shall be chosen large enough to reliably capture the maximum and minimum values of the vibration.

NOTE Large AC induction machines running at very low slip values at no load may require several minutes to more than ten minutes for such measurements to be completed at each vibration measuring position.

4.3 Relative shaft vibration

The criterion adopted for the relative shaft vibration shall be the vibratory displacement $s_{\text{p-p}}$ in the direction of measurement from~~ ISO 7919-1~~ ISO 20816-1.

5 Measurement equipment

The measurement equipment shall be capable of measuring broadband r.m.s. vibration with flat response over a frequency range of~~ at least~~ 10 Hz to 1 000 Hz, in accordance with the

requirements of ISO 2954. However, for machines with speeds approaching or below 600 min^{-1}, the lower limit of the flat response frequency range shall not be greater than 2 Hz.

Measurement equipment for relative shaft vibration measurements shall comply with the requirements in ISO 10817-1.

Multi-directional vibration sensors shall not be used.

NOTE Multi-directional sensors do not provide proper vibration measurement in all directions when mounted in only one location.

6 Machine mounting

6.1 General

The vibration of an electrical machine is closely linked with the mounting of the machine. To permit evaluation as far as balance and vibration of rotating electrical machines are concerned, it is necessary to measure the vibration on the machine alone, under properly determined test conditions, to enable reproducible tests to be carried out and to provide comparable measurements.

6.2 Free suspension

This condition is achieved by suspending the machine on a spring or by mounting on an elastic support (springs, rubber, etc.).

The highest natural oscillation frequency (f_{no}) of the suspension system and machine~~, in the six possible degrees of freedom~~, shall be less than 1/3 of the frequency f_1 corresponding to the speed of the machine under test, as defined in 7.3. Based on the mass of the machine being tested, the necessary elasticity of the suspension system as a function of ~~nominal~~ rated speed from 600 min^{-1} to 3 600 min^{-1} can be determined from Figure 1. For speeds lower than 600 min^{-1} measurements in free suspension are not practical. For speeds greater than 3 600 min^{-1}, the static displacement Z should be not less than the value for 3 600 min^{-1}.

The curve in Figure 1 presents the minimum elastic displacement to attain the necessary vertical rigid body natural oscillation, which is usually the highest rigid body natural frequency. Static displacement Z is expressed as:

$$Z = \frac{a^2 g}{(2\pi n)^2} \qquad , a = \frac{f_1}{f_{no}};\ a \geq 3$$

where

Z is the displacement in m,

n is the rated speed in units of s^{-1}, and

g is the acceleration of gravity (9,81 m/s^2).

When a is set to 3, then the curve in Figure 1 is generated.

6.3 Rigid mounting

6.3.1 Foundation

6.3.1.1 General

~~The maximum vibration velocity measured in the horizontal and vertical directions at the machine feet (or at the base frame near to the bearing pedestals or stator feet) shall not~~

~~exceed 25 % of the maximum velocity, which is measured at the adjacent bearing housing in the same measurement direction.~~

~~NOTE 1 This requirement ensures that the horizontal and vertical natural frequencies of the complete test arrangement do not coincide within:~~

~~a) ±10 % of the rotational frequency of the machine;~~

~~b) ± 5 % of twice the rotational frequency, or~~

~~c) ± 5 % of once and twice the electrical line frequency.~~

~~NOTE 2 The ratio of 25 % of foot to bearing vibration velocities is valid for once per revolution vibration and the vibration at twice the line frequency (if the latter is being evaluated).~~

During the shop running test of the assembled machine, vibration measurements shall be made with the machine properly shimmed and securely fastened to a massive foundation or test floor stand. Elastic mounts are not permitted.

The horizontal and vertical natural frequencies of the complete test arrangement shall not coincide within:

a) ± 10 % of the rotational frequency of the machine;

b) ± 5 % of twice the rotational frequency, or

c) ± 5 % of once and twice the electrical line frequency.

Either one of the following two mounting conditions may be chosen by the manufacturer.

6.3.1.2 Rigid mounting on massive foundation

One indication of massive foundation is when the vibration velocity measured in the horizontal and vertical directions at the machine feet (or at the base frame near to the bearing pedestals or stator feet) does not exceed 30 % of the maximum velocity, which is measured at the adjacent bearing housing in the same measurement direction. The ratio of foot to bearing vibration velocities is valid for the rotational frequency component or twice-line frequency component (if the latter is being evaluated).

NOTE 1 The rigidity of a foundation is a relative quantity. It is compared with the rigidity of the machine bearing system. The ratio of bearing housing vibration to foundation vibration is a characteristic quantity for the evaluation of foundation flexibility.

NOTE 2 If the machine is to be supported in the field by a structure other than a massive foundation, it may be necessary to perform a system dynamic analysis to make the necessary changes to the foundation dynamic stiffness.

6.3.1.3 Rigid mounting on test floor stand

This condition is achieved by mounting the machine on an adequately rigid test foundation free of resonances at forcing frequencies, see 6.3.1.1.

NOTE This mounting is the most used in manufacturers test labs.

6.3.2 Horizontal machines

~~The machine under test shall be:~~

- ~~clamped directly to a solid floor, or~~
- ~~through its mounting base plate to a solid floor, or~~
- ~~to a rigid plate that meets the requirements of 6.3.1.~~

The machine under test shall be bolted or clamped using all bolt-hole positions to a foundation that meets the requirements of 6.3.1.2 or 6.3.1.3.

There are constructions and mountings in which the above fixing conditions cannot be met, such as single-bearing machines. In those cases there should be an agreement between the supplier and customer.

6.3.3 Vertical machines

Vertical machines shall be mounted onto a solid rectangular or circular steel plate with a bore hole in the centre of the shaft extension, a machined surface for the flange of electrical machine being measured and ~~tapped holes for the flange bolts~~ holes provided for fasteners or clamps. The steel plate thickness shall be at least three times greater than the machine flange thickness, five times is ~~preferable~~ recommended. The edge length respective to the diameter shall be at least equal to the height of the top bearing, L. Figure 6 is an example for IM V1 (see IEC 60034-7).

The ~~mounting~~ steel base shall be clamped firmly and tilt safe to a solid floor and meet the requirements to 6.3.1.2 or 6.3.1.3. The flange connection shall use the correct number ~~and diameter~~ of fasteners or clamps. If the above method of mounting is not reasonable, other arrangements can be per agreement between supplier and customer.

6.4 Active environment determination

The support systems described in 6.2 and 6.3 are considered passive, admitting insignificant external disturbances to the machine. When, for the same measurement position, the vibration magnitude with the machine at standstill exceeds 25 % of the value when the machine is running, then an active environment is said to exist and this standard does not apply (see ~~ISO 10816-1~~ ISO 20816-1).

7 Conditions of measurement

7.1 Key

For the balancing and measurement of vibration on machines provided with a shaft extension keyway, the keyway shall ~~be filled with a half key,~~ be considered according to ~~ISO 8821~~ ISO 21940-32.

7.2 Measurement positions and directions

7.2.1 Measurement positions for vibration

The location of the preferred measurement positions and directions to which the levels of vibration magnitude apply are shown in Figure 2 for machines with end-shield bearings and in Figure 4 for machines with pedestal bearings. Figure 3 applies to those machines where measurement positions according to Figure 2 are not possible without disassembly of parts. When measurements cannot be performed per Figure 2 or Figure 3, there should be an agreement between supplier and customer.

Figure 6 applies to machines mounted in the vertical position.

NOTE 1 Measurement according to Figure 3 might be on the frame as close to the bearing housing as possible.

NOTE 2 Axial vibration measurement may not be possible without disassembly on both ends. If a machine has thrust bearings, this may be per agreement between supplier and customer, see 8.3.

7.2.2 Measurement positions for relative shaft displacement

Non-contacting transducers shall be installed inside the bearing, measuring directly the relative shaft journal displacement, or (when inside mounting is not possible) adjacent to the bearing shell. The preferred radial positions are as indicated in Figure 5.

7.3 Test conditions

Machines shall be tested at no-load with all relevant quantities at their rated value.

Fixed speed AC machines shall be run with a ~~virtually sinusoidal waveform~~ sinusoidal supply voltage according to IEC 60034-1.

The test shall be performed at each rated fixed speed or over the rated speed range for variable frequency drives. For all tested speeds, the values shall not exceed the corresponding limit in Table 1.

~~Machines intended for variable frequency drives shall be tested at the speeds for which the machine is intended during type tests.~~ In order to discern the mechanically induced vibration from other vibration excitation forces, it is recommended that DC machines are tested with a low current ripple supply or pure DC.

NOTE Tests with variable-frequency power supply normally only confirm mechanically induced vibrations. It is possible that electrically induced vibrations will be different. ~~To have complete factory tests it is necessary to test with the converter to be installed with the motor in situ.~~ If possible, the tests with the actual converter to be installed with the motor on site will provide better information about vibration behaviour.

For routine testing of variable-speed machines, it is permitted to test at a single speed based upon information obtained during type test.

For machines that are bi-directional, the vibration limits apply for both directions of rotation, but need to be measured in only one direction.

7.4 Vibration transducer

The mounting of the transducer used for vibration measurement ~~and~~ on the machine surface shall be as specified by the manufacturer of the transducer and shall not disturb the vibratory condition of the machine under test.

For this, it is necessary that the total coupled mass of the transducer assembly is less than 1/50 of the mass of the machine.

8 Limits of bearing housing vibration

8.1 Limits of vibration magnitude

The limits apply to the measured broadband r.m.s. vibration velocity and displacement ~~and acceleration~~ in the frequency range specified in Clause 5. ~~Decisive for the evaluation of vibration magnitude is the highest value of these three measurement quantities.~~

~~NOTE For routine tests of standard machines with speeds between 600 min^{-1} and 3 600 min^{-1} it is sufficient to measure only vibration velocity.~~

The vibration magnitude for DC and three phase AC machines with shaft heights 56 mm and higher ~~and~~, for one of either of the two mounting conditions according to Clause 6, shall not exceed the limits specified in Table 1. ~~Values~~ Limits are given for two vibration grades. When no grade is specified, machines complying with this standard shall be grade A.

For routine tests of standard machines with rotational speeds less than 600 min^{-1}, vibration is to be expressed in units of displacement. For rotational speeds from 600 min^{-1} up to 15 000 min^{-1}, vibration is to be expressed in units of velocity.

When the routine test is made with a free-suspension mounting condition, the type test should also include testing with rigid mounting. This is valid for the whole speed range of this standard.

Table 1 – Limits of maximum vibration magnitude in displacement (r.m.s.) and velocity (r.m.s.) ~~and acceleration (r.m.s.)~~ for shaft height *H*

~~Vibration grade~~	~~Shaft height, mm~~	~~$56 \leq H \leq 132$~~			~~$132 < H \leq 280$~~			~~$H > 280$~~		
	~~Mounting~~	~~Displac. µm~~	~~Vel. mm/s~~	~~Acc. m/s^2~~	~~Displac. µm~~	~~Vel. mm/s~~	~~Acc. m/s^2~~	~~Displac. µm~~	~~Vel. mm/s~~	~~Acc. m/s^2~~
~~A~~	~~Free suspension~~	~~25~~	~~1,6~~	~~2,5~~	~~35~~	~~2,2~~	~~3,5~~	~~45~~	~~2,8~~	~~4,4~~
	~~Rigid mounting~~	~~21~~	~~1,3~~	~~2,0~~	~~29~~	~~1,8~~	~~2,8~~	~~37~~	~~2,3~~	~~3,6~~
~~B~~	~~Free suspension~~	~~11~~	~~0,7~~	~~1,1~~	~~18~~	~~1,1~~	~~1,7~~	~~29~~	~~1,8~~	~~2,8~~
	~~Rigid mounting~~		~~–~~		~~14~~	~~0,9~~	~~1,4~~	~~24~~	~~1,5~~	~~2,4~~

Vibration grade	Shaft height, mm	$56 \leq H \leq 132$		$H > 132$	
	Mounting	Displacement µm	Velocity mm/s	Displacement µm	Velocity mm/s
A	Free suspension	45	2,8	45	2,8
	Rigid mounting	–	–	37	2,3 2,8*
B	Free suspension	18	1,1	29	1,8
	Rigid mounting	–	–	24	1,5 1,8*

Grade A applies to machines with no special vibration requirements.

Grade B applies to machines with special vibration requirements.

Rigid mounting is not considered acceptable for machines with shaft heights less than or equal to 132 mm.

~~The interface frequencies for displacement/velocity and velocity/acceleration are 10 Hz and 250 Hz respectively.~~

Vibration at frequencies above 1 000 Hz should be filtered out.

The shaft height of a machine without feet, or a machine with raised feet, or any vertical machine is to be taken as the shaft height of a machine in the same basic frame, but of the horizontal shaft foot-mounting type.

* This level is the limit when the twice line frequency vibration level is dominant as defined in 8.2 and explained in Figure 7.

NOTE 1 The manufacturer and the purchaser ~~should~~ take into account that the instrumentation can have a measurement tolerance of ± 10 %.

~~NOTE The shaft height of a machine without feet, or a machine with raised feet, or any vertical machine is to be taken as the shaft height of a machine in the same basic frame, but of the horizontal shaft foot-mounting type.~~

NOTE 2 A machine which is well-balanced in itself and of a grade conforming with Table 1 may exhibit large vibrations when installed ~~in situ~~ on site arising from various causes, such as unsuitable foundations, reaction of the driven machine, current ripple from the power supply, etc. Vibration may also be caused by driving elements with a natural oscillation frequency very close to the excitation due to the small residual unbalance of the rotating masses of the machine. In such cases, checks ~~should~~ can be carried out on site not only on the machine, but also on each element of the installation. (See ISO 10816-3.)

NOTE 3 As mentioned for onsite operation in ISO 20816-1, special agreements can be made. The values given in ISO 20816-1 are the basis for the ease of discussion and agreement between the supplier and the user. The values ensure in most applications that major errors or unrealistic requirements are avoided. Special agreements are also made for acceleration and deceleration ramps. Short term vibration limits and reduced bearing life time as a function of vibration velocity can be given by the manufacturer.

8.2 Limits of vibration velocity with twice-line frequency for a.c. machines

Two-pole induction machines may have electromagnetic excited vibration at twice the frequency of the power system. The correct evaluation of these vibration components requires a rigid mounting of the machine, which complies with the requirements given in 6.3.

When type tests demonstrate a dominant twice line frequency component for machines having H > ~~280~~ 132 mm, the vibration magnitude limit in Table 1 (for Grade A) is increased to 2,8 mm/s (r.m.s.) from 2,3 mm/s (r.m.s.) or (for Grade B) is increased to 1,8 mm/s (r.m.s.) from 1,5 mm/s (r.m.s.). Greater values are subject to prior agreement. A twice line frequency component is considered dominant when type tests demonstrate it to be greater than 70 % of 2,3 mm/s (r.m.s.) (for Grade A) or 70 % of 1,5 mm/s (r.m.s.) (for Grade B). Graphical interpretation of the definition above is shown in Figure 7.

NOTE 1 With respect to the twice-line frequency component agreements between the supplier and the customer can be made.

NOTE 2 70 % approximates $\frac{1}{\sqrt{2}}$ 100 %, where the rms values of rotational and twice-line frequency are equal.

8.3 Axial vibration

The evaluation of axial bearing vibration depends on the bearing function and bearing construction.

In the case of thrust bearings, axial vibration correlates with thrust pulsations, which could cause damage to the metal liner of sleeve bearings or to parts of antifriction bearings. Axial vibration of these bearings shall be evaluated in the same manner as transverse vibration and Table 1 limits apply.

For bearings having no axial limiting construction such as sleeve bearings with no axial thrust bearing, the requirement may be relaxed by prior agreement.

9 Limits of relative shaft vibration

Relative shaft vibration measurements are recommended only for sleeve bearing machines with speeds >1 200 min^{-1}; rated power >1 000 kW; and shall be subject to prior agreement with respect to the necessary provisions regarding the installation of shaft measurement transducers.

When machines with sleeve bearings have special provisions for the installation of shaft vibration measurement transducers, the limits for the relative shaft vibration displacement are specified in Table 2. These limits are in addition to those required in Clause 8.

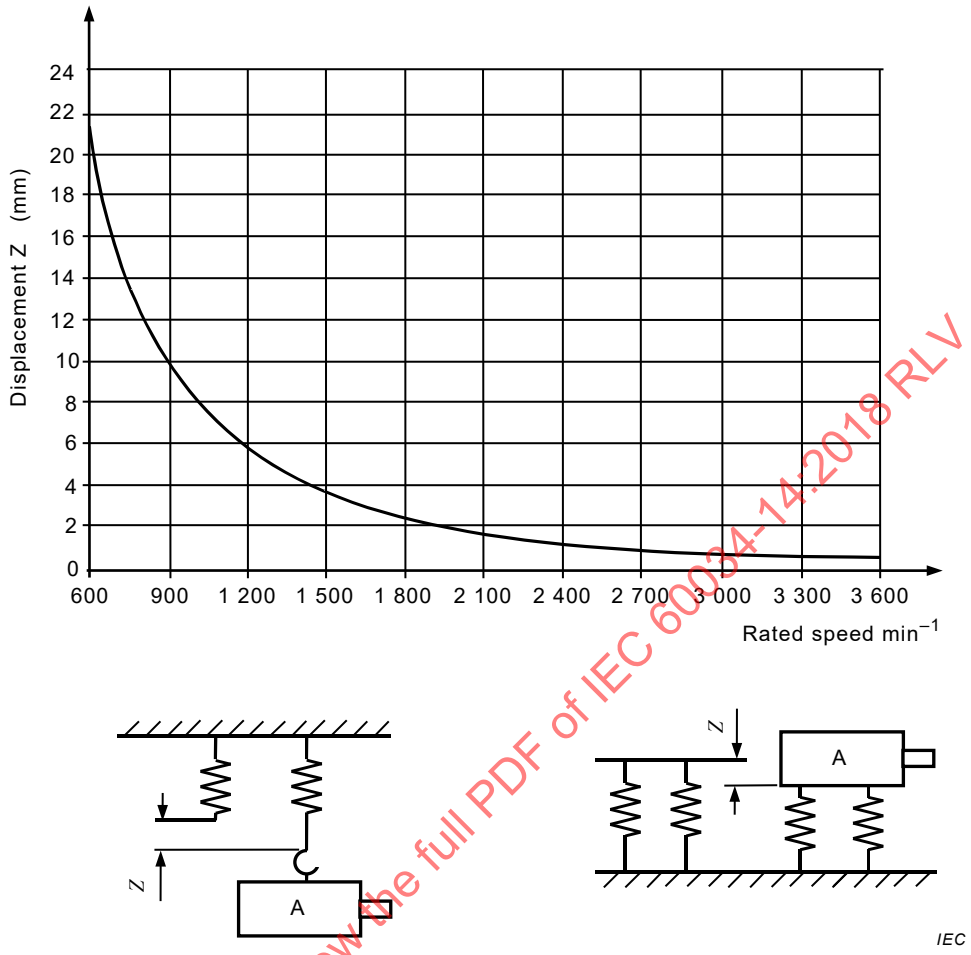
Table 2 – Limits for the maximum shaft vibration (S_{p-p}) and the maximum run-out

Vibration grade	Speed range min^{-1}	Maximum relative shaft displacement µm	Maximum combined mechanical and electrical run-out µm
A	> 1 800	65	16
	≤ 1 800	90	23
B	> 1 800	50	12,5
	≤ 1 800	65	16

~~NOTE 1 machines with vibration grade "B" are frequently specified for high speed drives in critical installations.~~

Grade B applies to machines with special vibration requirements.

~~NOTE 2~~ The maximum relative shaft displacement limits include the run-out. For the definition of the run-out see ~~ISO 7919-1~~ ISO 20816-1.



Key

A machine under test

Z displacement

Figure 1 – Minimum elastic displacement as a function of rated speed

~~NOTE~~ To reduce the influence of the mass and the moments of inertia of the suspension system on the vibration level, the effective mass of the elastic support ~~should~~ shall be not greater than 1/10 of that of the machine.

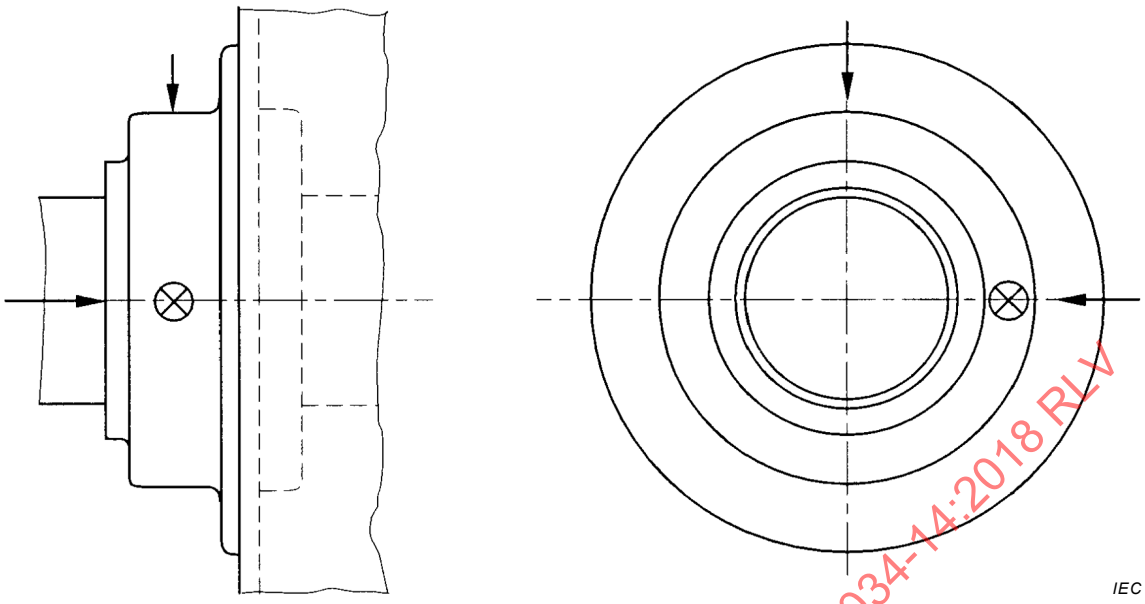


Figure 2 – Preferred positions of measurement applicable to one or both ends of the machine

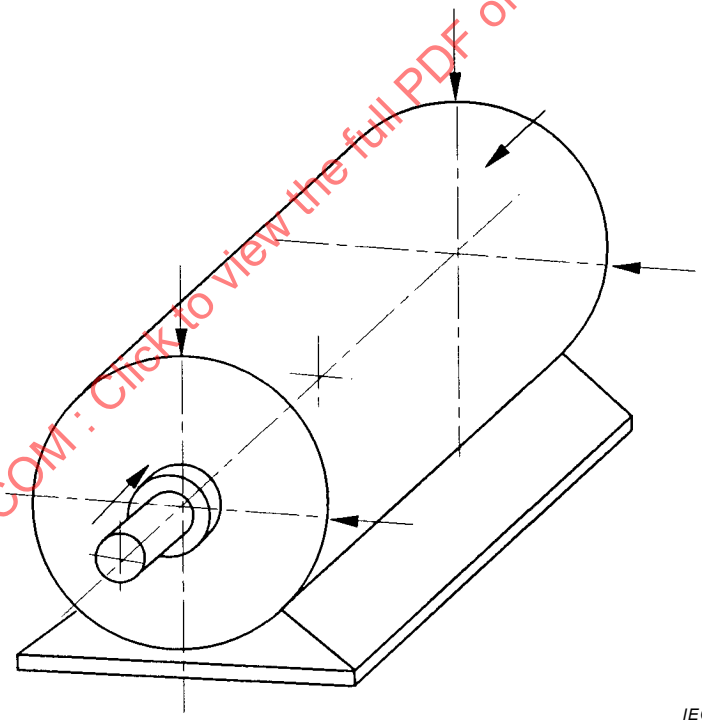


Figure 3 – Measurement positions for those ends of machines where measurements according to Figure 2 are not possible without disassembly of parts

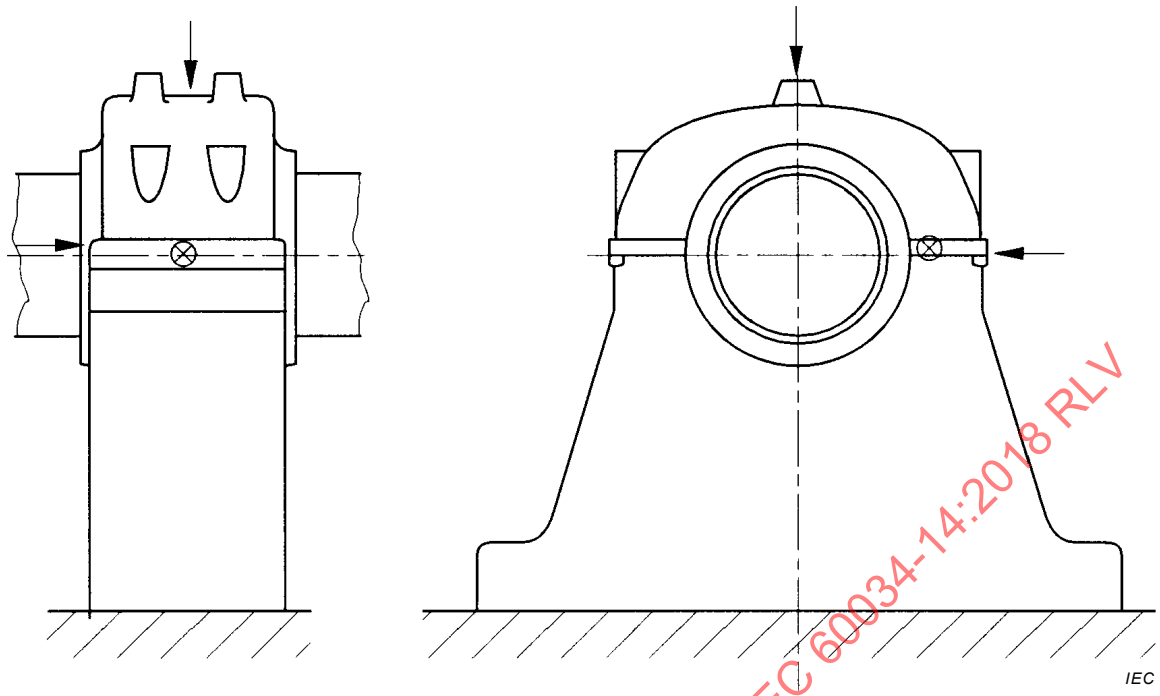
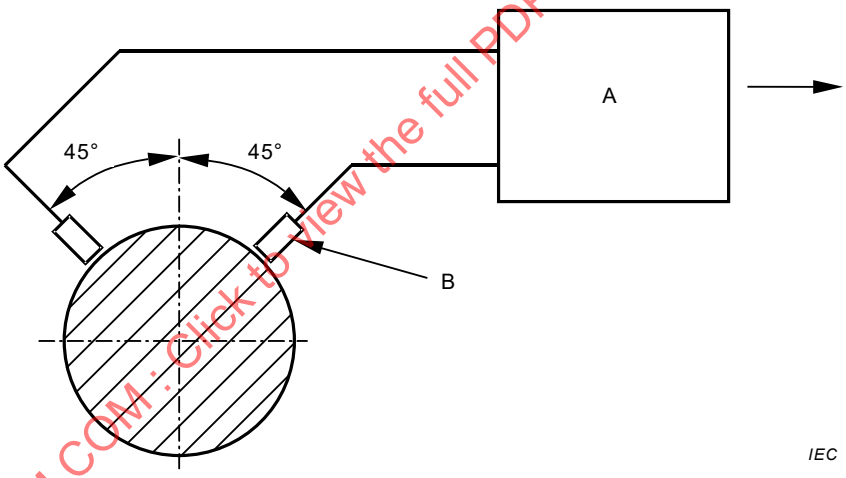


Figure 4 – Measurement positions for pedestal bearing



Key

A signal conditioner

B transducer

Figure 5 – Preferred circumferential position of transducers for the measurement of relative shaft displacement

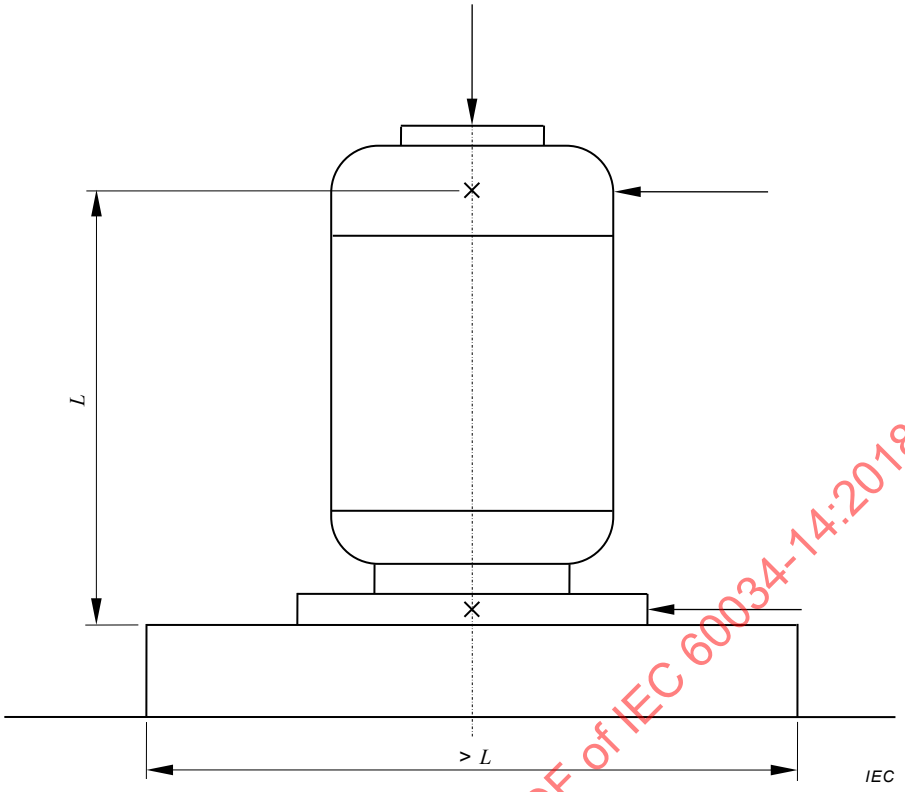


Figure 6 – Measurement positions for vertical machines (measurements to be made at the bearing housing; when not accessible, then as close as possible)

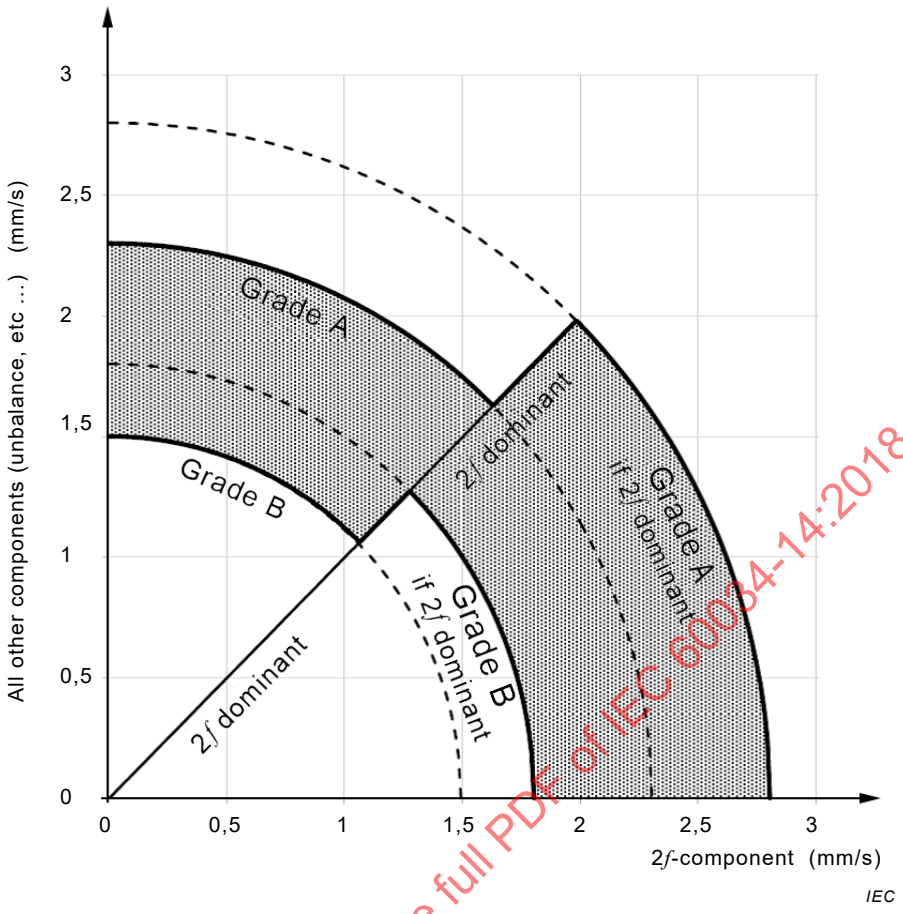


Figure 7 – Vibration limit diagram for shaft heights H > 132 mm

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API Standard 547:2017, *General Purpose Form-wound Squirrel Cage Induction Motors – 185 kW (250 hp) and through 2240 kW (3 000 hp)*

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NEMA MG 1:2016, *Motors and Generators – Part 7: Mechanical vibration measurement, evaluation and limits*

IEC 60034-14

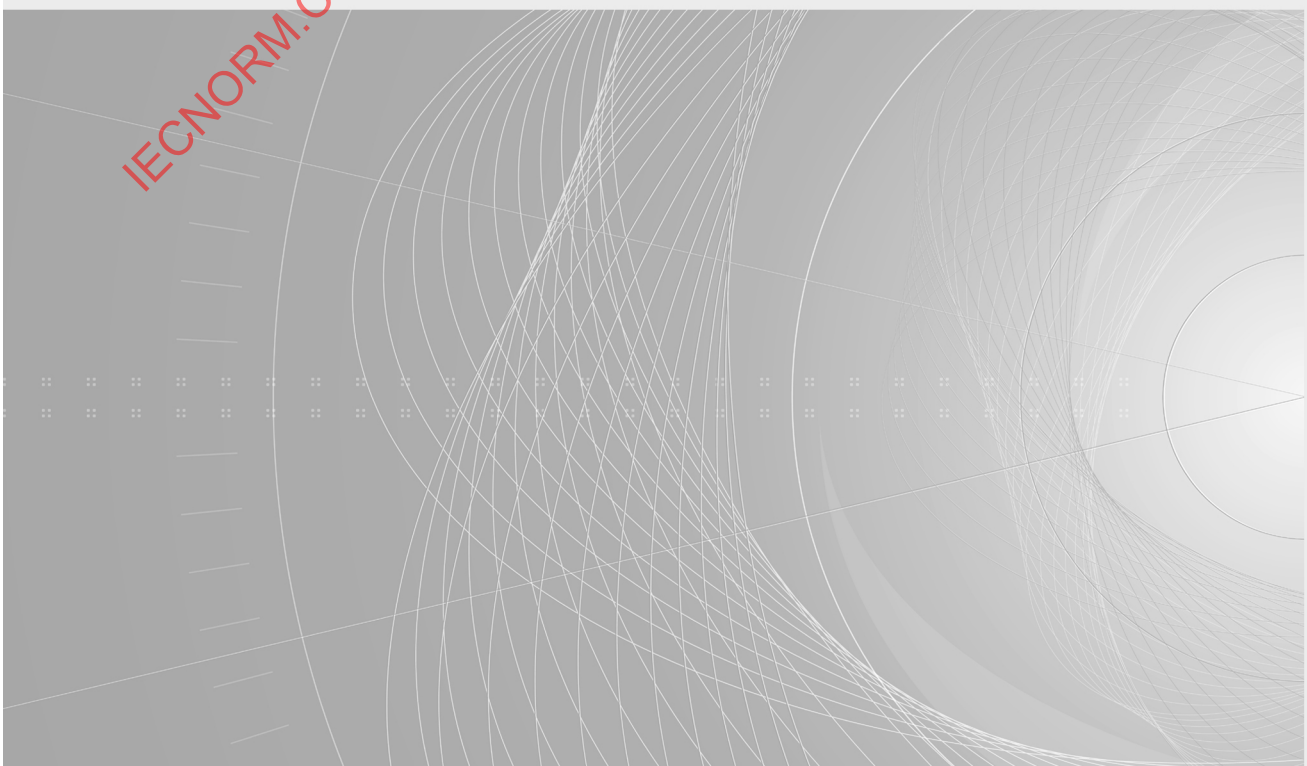
Edition 4.0 2018-08

INTERNATIONAL STANDARD

NORME INTERNATIONALE

Rotating electrical machines – Part 14: Mechanical vibration of certain machines with shaft heights 56 mm and higher – Measurement, evaluation and limits of vibration severity

Machines électriques tournantes – Partie 14: Vibrations mécaniques de certaines machines de hauteur d'axe supérieure ou égale à 56 mm – Mesurage, évaluation et limites de l'intensité vibratoire



IEC 60034-14:2018-08(en-fr)

CONTENTS

INTERNATIONAL ELECTROTECHNICAL COMMISSION

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ROTATING ELECTRICAL MACHINES –

Part 14: Mechanical vibration of certain machines with shaft heights 56 mm and higher – Measurement, evaluation and limits of vibration severity

FOREWORD

1) The International Electrotechnical Commission (IEC) is a worldwide organization for standardization comprising all national electrotechnical committees (IEC National Committees). The object of IEC is to promote international co-operation on all questions concerning standardization in the electrical and electronic fields. To this end and in addition to other activities, IEC publishes International Standards, Technical Specifications, Technical Reports, Publicly Available Specifications (PAS) and Guides (hereafter referred to as "IEC Publication(s)"). Their preparation is entrusted to technical committees; any IEC National Committee interested in the subject dealt with may participate in this preparatory work. International, governmental and non-governmental organizations liaising with the IEC also participate in this preparation. IEC collaborates closely with the International Organization for Standardization (ISO) in accordance with conditions determined by agreement between the two organizations.
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International Standard IEC 60034-14 has been prepared by IEC technical committee 2: Rotating machinery.

This fourth edition cancels and replaces the third edition, published in 2003, and its amendment 1, published in 2007. It constitutes a technical revision.

The significant technical changes with respect to the previous edition are:

a) 6.2 is significantly changed to better explain the definition "free suspension".
b) 6.3: a second method of rigid mount is added since the first method is not always possible on the test floor.
c) 7.1: an improved option for shaft key is defined.

d) Clause 8: considerable effort to harmonize with NEMA MG 1 and IEEE 841 and API 541, and also establish levels which are achievable and more in line with best practices. Table 1 is reduced to two shaft-height range sections.

e) 8.2: definition of twice line frequency simplified along with Figure 7 added.

The text of this International Standard is based on the following documents:

FDIS	Report on voting
2/1906/FDIS	2/1914/RVD

Full information on the voting for the approval of this International Standard can be found in the report on voting indicated in the above table.

This document has been drafted in accordance with the ISO/IEC Directives, Part 2.

A list of all parts in the IEC 60034 series, published under the general title *Rotating electrical machines*, can be found on the IEC website.

NOTE For A table of cross-references of all IEC TC 2 publications can be found in the IEC TC 2 dashboard on the IEC website.

The committee has decided that the contents of this document will remain unchanged until the stability date indicated on the IEC website under "http://webstore.iec.ch" in the data related to the specific document. At this date, the document will be

- reconfirmed,
- withdrawn,
- replaced by a revised edition, or
- amended.

ROTATING ELECTRICAL MACHINES –

Part 14: Mechanical vibration of certain machines with shaft heights 56 mm and higher – Measurement, evaluation and limits of vibration severity

1 Scope

This part of IEC 60034 specifies the factory acceptance vibration test procedures and vibration limits for certain electrical machines under specified conditions, when uncoupled from any load or prime mover.

It is applicable to DC and three-phase AC machines, with shaft heights 56 mm and higher and a rated output up to 50 MW, at operational speeds from 120 min^{-1} up to and including 15 000 min^{-1}.

This document is not applicable to machines mounted *in situ* (on site), three-phase commutator motors, single-phase machines, three-phase machines operated on single-phase systems, vertical waterpower generators, turbine generators greater than 20 MW and machines with magnetic bearings or series-wound machines.

NOTE For machines measured *in situ,* refer to applicable parts of ISO 20816, ISO 10816 and ISO 7919.

2 Normative references

The following documents are referred to in the text in such a way that some or all of their content constitutes requirements of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

IEC 60034-1, *Rotating electrical machines – Part 1: Rating and performance*

IEC 60034-7, *Rotating electrical machines – Part 7: Classification of types of constructions and mounting arrangements (IM Code)*

ISO 2954, *Mechanical vibration of rotating and reciprocating machinery – Requirements for instruments for measuring vibration severity*

ISO 10817-1, *Rotating shaft vibration measuring systems – Part 1: Relative and absolute sensing of radial vibration from rotating shafts*

ISO 20816-1, *Mechanical vibration – Measurement and evaluation of machine vibration – Part 1: General guidelines*

ISO 21940-32, *Mechanical vibration – Rotor balancing – Part 32: Shaft and fitment key convention*

3 Terms and definitions

For the purposes of this document, the terms and definitions given in IEC 60034-1 apply.

ISO and IEC maintain terminological databases for use in standardization at the following addresses:

- IEC Electropedia: available at http://www.electropedia.org/
- ISO Online browsing platform: available at http://www.iso.org/obp

4 Measurement quantities

4.1 General

Measurement quantities are the vibration displacement and vibration velocity at the machine bearings and the relative shaft vibration displacement within or near to the machine bearings.

4.2 Vibration magnitude

The criterion for the vibration magnitude at the machine bearings shall be the broadband r.m.s. value of the vibration displacement in micrometres or the vibration velocity in millimeters per second in the frequency range specified in Clause 5. The maximum value, determined at the prescribed measurement positions and prescribed measuring variable, according to this standard, characterizes the vibration magnitude of the machine.

Induction motors (especially of two pole type) frequently show vibration beating at twice slip frequency. In these cases, the decisive vibration magnitude shall be determined from the relationship:

$$x_{\text{r.m.s.}} = \sqrt{\frac{x_{\text{max}}^{\ 2} + x_{\text{min}}^{\ 2}}{2}}$$

where

x_{max} is the maximum r.m.s. value of vibration displacement or vibration velocity;

x_{min} is the minimum r.m.s. value of vibration displacement or vibration velocity.

The sample rate shall be chosen large enough to reliably capture the maximum and minimum values of the vibration.

NOTE Large AC induction machines running at very low slip values at no load may require several minutes to more than ten minutes for such measurements to be completed at each vibration measuring position.

4.3 Relative shaft vibration

The criterion adopted for the relative shaft vibration shall be the vibratory displacement $s_{\text{p-p}}$ in the direction of measurement from ISO 20816-1.

5 Measurement equipment

The measurement equipment shall be capable of measuring broadband r.m.s. vibration with flat response over a frequency range of 10 Hz to 1 000 Hz, in accordance with the requirements of ISO 2954. However, for machines with speeds approaching or below 600 min^{-1}, the lower limit of the flat response frequency range shall not be greater than 2 Hz.

Measurement equipment for relative shaft vibration measurements shall comply with the requirements in ISO 10817-1.

Multi-directional vibration sensors shall not be used.

NOTE Multi-directional sensors do not provide proper vibration measurement in all directions when mounted in only one location.

6 Machine mounting

6.1 General

The vibration of an electrical machine is closely linked with the mounting of the machine. To permit evaluation as far as balance and vibration of rotating electrical machines are concerned, it is necessary to measure the vibration on the machine alone, under properly determined test conditions, to enable reproducible tests to be carried out and to provide comparable measurements.

6.2 Free suspension

This condition is achieved by suspending the machine on a spring or by mounting on an elastic support (springs, rubber, etc.).

The highest natural oscillation frequency (f_{no}) of the suspension system and machine, shall be less than 1/3 of the frequency f_1 corresponding to the speed of the machine under test, as defined in 7.3. Based on the mass of the machine being tested, the necessary elasticity of the suspension system as a function of rated speed from 600 min^{-1} to 3 600 min^{-1} can be determined from Figure 1. For speeds lower than 600 min^{-1} measurements in free suspension are not practical. For speeds greater than 3 600 min^{-1}, the static displacement Z should be not less than the value for 3 600 min^{-1}.

The curve in Figure 1 presents the minimum elastic displacement to attain the necessary vertical rigid body natural oscillation, which is usually the highest rigid body natural frequency. Static displacement Z is expressed as:

$$Z = \frac{a^2 g}{(2\pi n)^2} \qquad , a = \frac{f_1}{f_{no}};\ a \geq 3$$

where

Z is the displacement in m,

n is the rated speed in units of s^{-1}, and

g is the acceleration of gravity (9,81 m/s^2).

When a is set to 3, then the curve in Figure 1 is generated.

6.3 Rigid mounting

6.3.1 Foundation

6.3.1.1 General

During the shop running test of the assembled machine, vibration measurements shall be made with the machine properly shimmed and securely fastened to a massive foundation or test floor stand. Elastic mounts are not permitted.

The horizontal and vertical natural frequencies of the complete test arrangement shall not coincide within:

a) ± 10 % of the rotational frequency of the machine;

b) ± 5 % of twice the rotational frequency, or

c) ± 5 % of once and twice the electrical line frequency.

Either one of the following two mounting conditions may be chosen by the manufacturer.

6.3.1.2 Rigid mounting on massive foundation

One indication of massive foundation is when the vibration velocity measured in the horizontal and vertical directions at the machine feet (or at the base frame near to the bearing pedestals or stator feet) does not exceed 30 % of the maximum velocity, which is measured at the adjacent bearing housing in the same measurement direction. The ratio of foot to bearing vibration velocities is valid for the rotational frequency component or twice-line frequency component (if the latter is being evaluated).

NOTE 1 The rigidity of a foundation is a relative quantity. It is compared with the rigidity of the machine bearing system. The ratio of bearing housing vibration to foundation vibration is a characteristic quantity for the evaluation of foundation flexibility.

NOTE 2 If the machine is to be supported in the field by a structure other than a massive foundation, it may be necessary to perform a system dynamic analysis to make the necessary changes to the foundation dynamic stiffness.

6.3.1.3 Rigid mounting on test floor stand

This condition is achieved by mounting the machine on an adequately rigid test foundation free of resonances at forcing frequencies, see 6.3.1.1.

NOTE This mounting is the most used in manufacturers test labs.

6.3.2 Horizontal machines

The machine under test shall be bolted or clamped using all bolt-hole positions to a foundation that meets the requirements of 6.3.1.2 or 6.3.1.3.

There are constructions and mountings in which the above fixing conditions cannot be met, such as single-bearing machines. In those cases there should be an agreement between the supplier and customer.

6.3.3 Vertical machines

Vertical machines shall be mounted onto a solid rectangular or circular steel plate with a bore hole in the centre of the shaft extension, a machined surface for the flange of electrical machine being measured and holes provided for fasteners or clamps. The steel plate thickness shall be at least three times greater than the machine flange thickness, five times is recommended. The edge length respective to the diameter shall be at least equal to the height of the top bearing, L. Figure 6 is an example for IM V1 (see IEC 60034-7).

The steel base shall be clamped firmly and tilt safe to a solid floor and meet the requirements to 6.3.1.2 or 6.3.1.3. The flange connection shall use the correct number of fasteners or clamps. If the above method of mounting is not reasonable, other arrangements can be per agreement between supplier and customer.

6.4 Active environment determination

The support systems described in 6.2 and 6.3 are considered passive, admitting insignificant external disturbances to the machine. When, for the same measurement position, the vibration magnitude with the machine at standstill exceeds 25 % of the value when the machine is running, then an active environment is said to exist and this standard does not apply (see ISO 20816-1).

7 Conditions of measurement

7.1 Key

For the balancing and measurement of vibration on machines provided with a shaft extension keyway, the keyway shall be considered according to ISO 21940-32.

7.2 Measurement positions and directions

7.2.1 Measurement positions for vibration

The location of the preferred measurement positions and directions to which the levels of vibration magnitude apply are shown in Figure 2 for machines with end-shield bearings and in Figure 4 for machines with pedestal bearings. Figure 3 applies to those machines where measurement positions according to Figure 2 are not possible without disassembly of parts. When measurements cannot be performed per Figure 2 or Figure 3, there should be an agreement between supplier and customer.

Figure 6 applies to machines mounted in the vertical position.

NOTE 1 Measurement according to Figure 3 might be on the frame as close to the bearing housing as possible.

NOTE 2 Axial vibration measurement may not be possible without disassembly on both ends. If a machine has thrust bearings, this may be per agreement between supplier and customer, see 8.3.

7.2.2 Measurement positions for relative shaft displacement

Non-contacting transducers shall be installed inside the bearing, measuring directly the relative shaft journal displacement, or (when inside mounting is not possible) adjacent to the bearing shell. The preferred radial positions are as indicated in Figure 5.

7.3 Test conditions

Machines shall be tested at no-load with all relevant quantities at their rated value.

Fixed speed AC machines shall be run with a sinusoidal supply voltage according to IEC 60034-1.

The test shall be performed at each rated fixed speed or over the rated speed range for variable frequency drives. For all tested speeds, the values shall not exceed the corresponding limit in Table 1.

In order to discern the mechanically induced vibration from other vibration excitation forces, it is recommended that DC machines are tested with a low current ripple supply or pure DC.

NOTE Tests with variable-frequency power supply normally only confirm mechanically induced vibrations. It is possible that electrically induced vibrations will be different. If possible, the tests with the actual converter to be installed with the motor on site will provide better information about vibration behaviour.

For routine testing of variable-speed machines, it is permitted to test at a single speed based upon information obtained during type test.

For machines that are bi-directional, the vibration limits apply for both directions of rotation, but need to be measured in only one direction.

7.4 Vibration transducer

The mounting of the transducer used for vibration measurement on the machine surface shall be as specified by the manufacturer of the transducer and shall not disturb the vibratory condition of the machine under test.

For this, it is necessary that the total coupled mass of the transducer assembly is less than 1/50 of the mass of the machine.

8 Limits of bearing housing vibration

8.1 Limits of vibration magnitude

The limits apply to the measured broadband r.m.s. vibration velocity and displacement in the frequency range specified in Clause 5.

The vibration magnitude for DC and three phase AC machines with shaft heights 56 mm and higher, for one of either of the two mounting conditions according to Clause 6, shall not exceed the limits specified in Table 1. Limits are given for two vibration grades. When no grade is specified, machines complying with this standard shall be grade A.

For routine tests of standard machines with rotational speeds less than 600 min^{-1}, vibration is to be expressed in units of displacement. For rotational speeds from 600 min^{-1} up to 15 000 min^{-1}, vibration is to be expressed in units of velocity.

When the routine test is made with a free-suspension mounting condition, the type test should also include testing with rigid mounting. This is valid for the whole speed range of this standard.

Table 1 – Limits of maximum vibration magnitude in displacement (r.m.s.) and velocity (r.m.s.) for shaft height H

Vibration grade	Shaft height, mm	$56 \leq H \leq 132$		$H > 132$	
	Mounting	Displacement µm	Velocity mm/s	Displacement µm	Velocity mm/s
A	Free suspension	45	2,8	45	2,8
	Rigid mounting	–	–	37	2,3 2,8*
B	Free suspension	18	1,1	29	1,8
	Rigid mounting	–	–	24	1,5 1,8*

Grade A applies to machines with no special vibration requirements.

Grade B applies to machines with special vibration requirements.

Rigid mounting is not considered acceptable for machines with shaft heights less than or equal to 132 mm.

Vibration at frequencies above 1 000 Hz should be filtered out.

The shaft height of a machine without feet, or a machine with raised feet, or any vertical machine is to be taken as the shaft height of a machine in the same basic frame, but of the horizontal shaft foot-mounting type.

* This level is the limit when the twice line frequency vibration level is dominant as defined in 8.2 and explained in Figure 7.

NOTE 1 The manufacturer and the purchaser take into account that the instrumentation can have a measurement tolerance of ± 10 %.

NOTE 2 A machine which is well-balanced in itself and of a grade conforming with Table 1 may exhibit large vibrations when installed on site arising from various causes, such as unsuitable foundations, reaction of the driven machine, current ripple from the power supply, etc. Vibration may also be caused by driving elements with a natural oscillation frequency very close to the excitation due to the small residual unbalance of the rotating masses

of the machine. In such cases, checks can be carried out on site not only on the machine, but also on each element of the installation. (See ISO 10816-3.)

NOTE 3 As mentioned for onsite operation in ISO 20816-1, special agreements can be made. The values given in ISO 20816-1 are the basis for the ease of discussion and agreement between the supplier and the user. The values ensure in most applications that major errors or unrealistic requirements are avoided. Special agreements are also made for acceleration and deceleration ramps. Short term vibration limits and reduced bearing life time as a function of vibration velocity can be given by the manufacturer.

8.2 Limits of vibration velocity with twice-line frequency for a.c. machines

Two-pole induction machines may have electromagnetic excited vibration at twice the frequency of the power system. The correct evaluation of these vibration components requires a rigid mounting of the machine, which complies with the requirements given in 6.3.

When type tests demonstrate a dominant twice line frequency component for machines having H > 132 mm, the vibration magnitude limit in Table 1 (for Grade A) is increased to 2,8 mm/s (r.m.s.) from 2,3 mm/s (r.m.s.) or (for Grade B) is increased to 1,8 mm/s (r.m.s.) from 1,5 mm/s (r.m.s.). Greater values are subject to prior agreement. A twice line frequency component is considered dominant when type tests demonstrate it to be greater than 70 % of 2,3 mm/s (r.m.s.)(for Grade A) or 70 % of 1,5 mm/s (r.m.s.) (for Grade B). Graphical interpretation of the definition above is shown in Figure 7.

NOTE 1 With respect to the twice-line frequency component agreements between the supplier and the customer can be made.

NOTE 2 70 % approximates $\frac{1}{\sqrt{2}}100\,\%$, where the rms values of rotational and twice-line frequency are equal.

8.3 Axial vibration

The evaluation of axial bearing vibration depends on the bearing function and bearing construction.

In the case of thrust bearings, axial vibration correlates with thrust pulsations, which could cause damage to the metal liner of sleeve bearings or to parts of antifriction bearings. Axial vibration of these bearings shall be evaluated in the same manner as transverse vibration and Table 1 limits apply.

For bearings having no axial limiting construction such as sleeve bearings with no axial thrust bearing, the requirement may be relaxed by prior agreement.

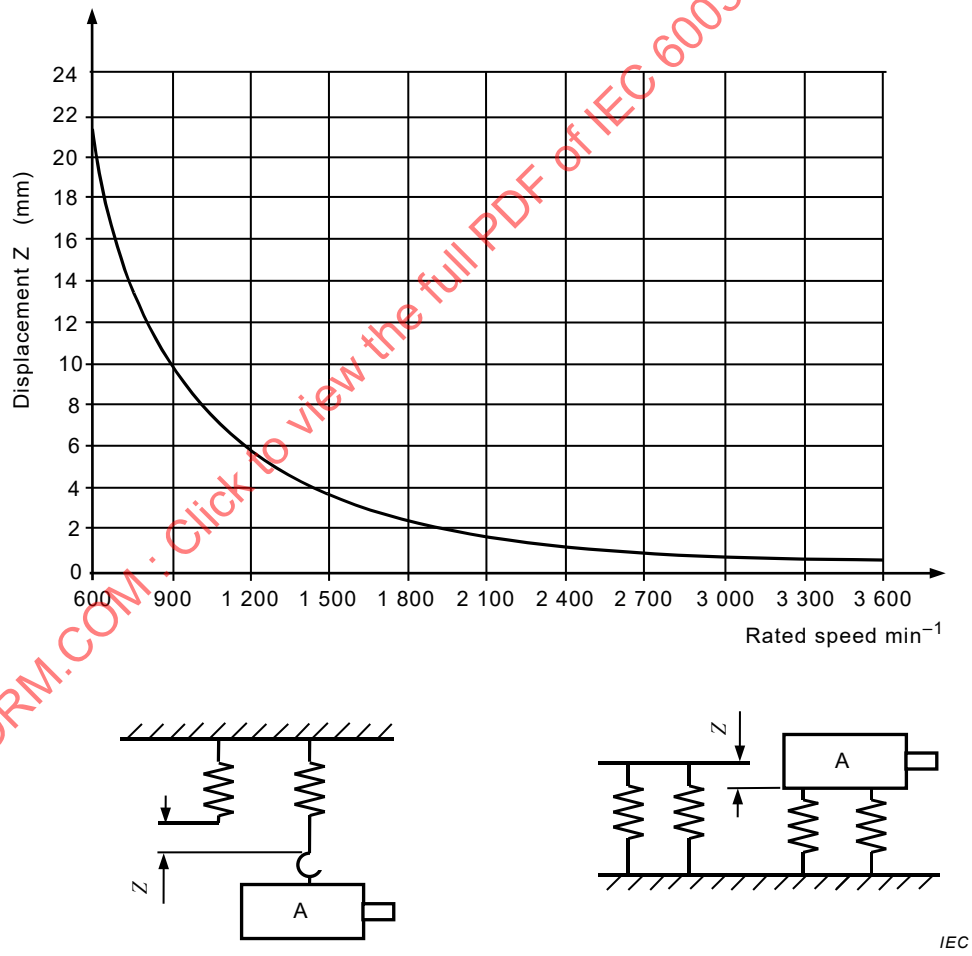
9 Limits of relative shaft vibration

Relative shaft vibration measurements are recommended only for sleeve bearing machines with speeds >1 200 min^{-1}; rated power >1 000 kW; and shall be subject to prior agreement with respect to the necessary provisions regarding the installation of shaft measurement transducers.

When machines with sleeve bearings have special provisions for the installation of shaft vibration measurement transducers, the limits for the relative shaft vibration displacement are specified in Table 2. These limits are in addition to those required in Clause 8.

Table 2 – Limits for the maximum shaft vibration (S_{p-p}) and the maximum run-out

Vibration grade	Speed range min^{-1}	Maximum relative shaft displacement µm	Maximum combined mechanical and electrical run-out µm
A	> 1 800	65	16
	≤ 1 800	90	23
B	> 1 800	50	12,5
	≤ 1 800	65	16
Grade B applies to machines with special vibration requirements. The maximum relative shaft displacement limits include the run-out. For the definition of the run-out see ISO 20816-1.			



Key

A machine under test

Z displacement

Figure 1 – Minimum elastic displacement as a function of rated speed

To reduce the influence of the mass and the moments of inertia of the suspension system on the vibration level, the effective mass of the elastic support shall be not greater than 1/10 of that of the machine.

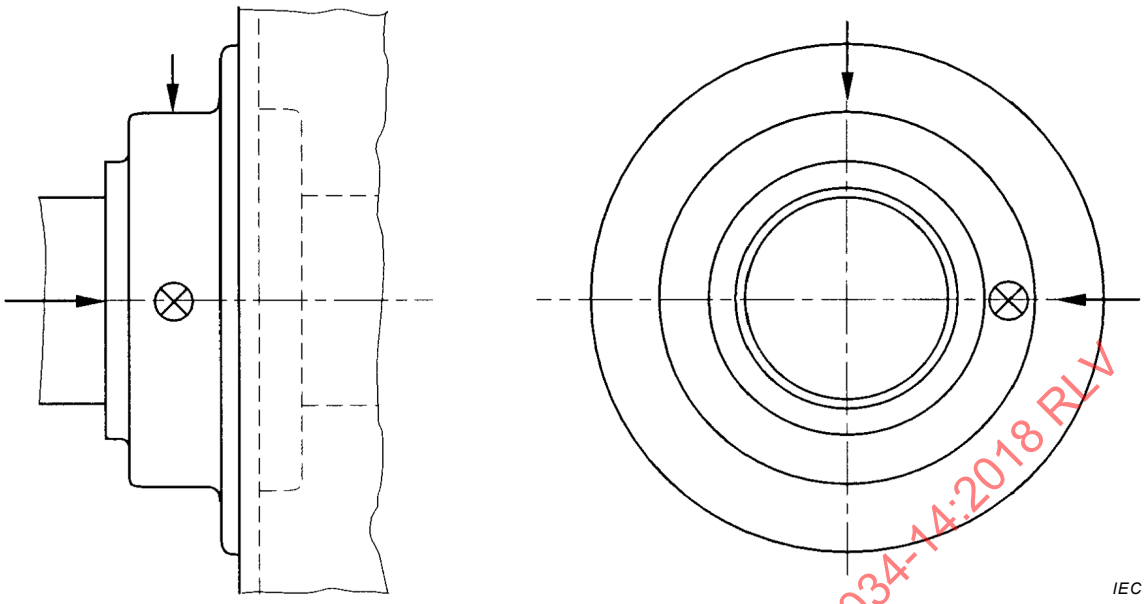


Figure 2 – Preferred positions of measurement applicable to one or both ends of the machine

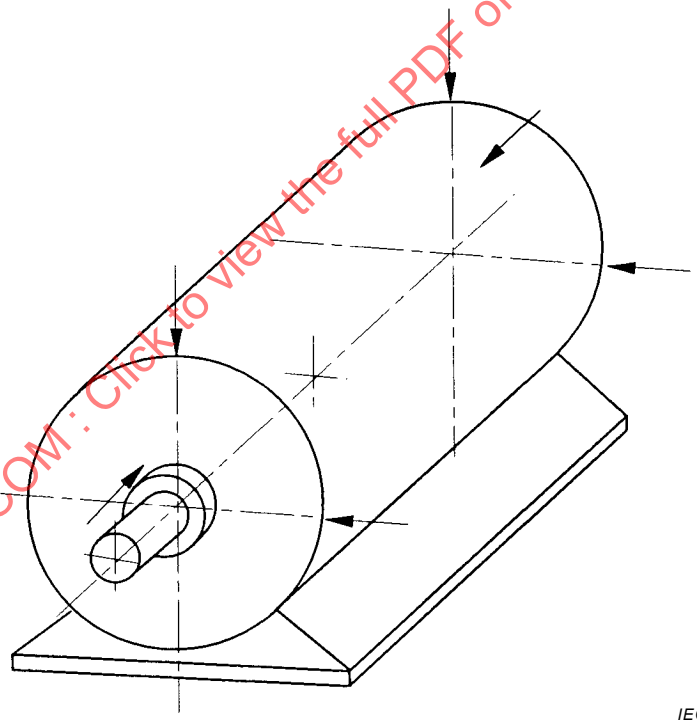


Figure 3 – Measurement positions for those ends of machines where measurements according to Figure 2 are not possible without disassembly of parts

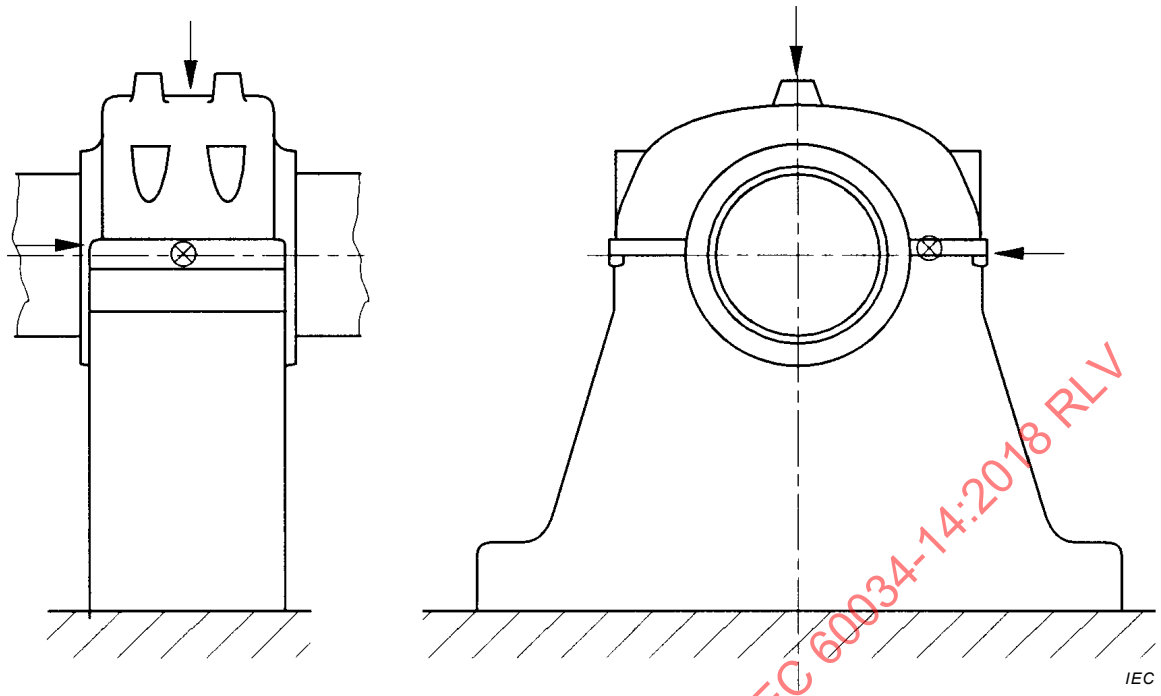
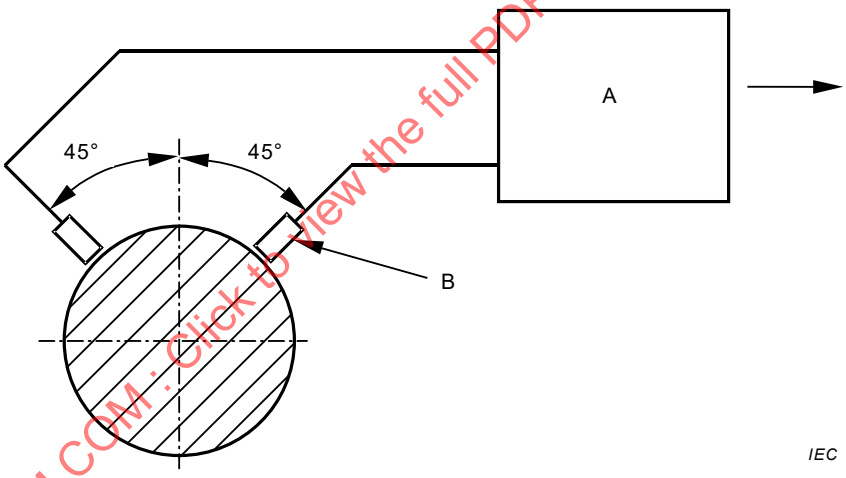


Figure 4 – Measurement positions for pedestal bearing



Key

A signal conditioner

B transducer

Figure 5 – Preferred circumferential position of transducers for the measurement of relative shaft displacement

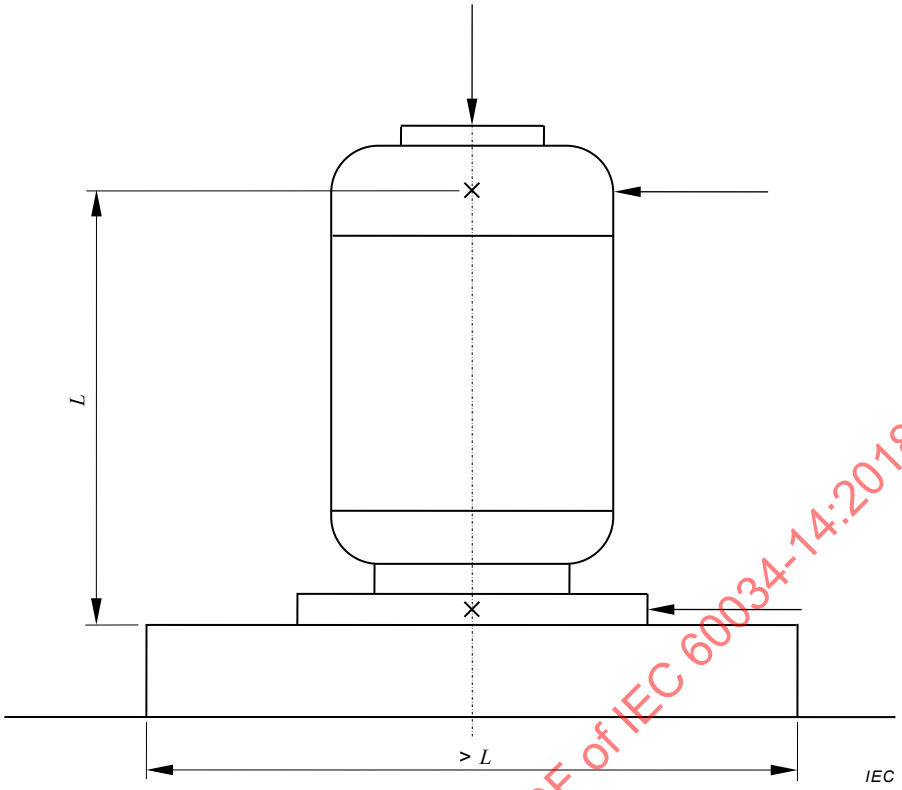


Figure 6 – Measurement positions for vertical machines (measurements to be made at the bearing housing; when not accessible, then as close as possible)

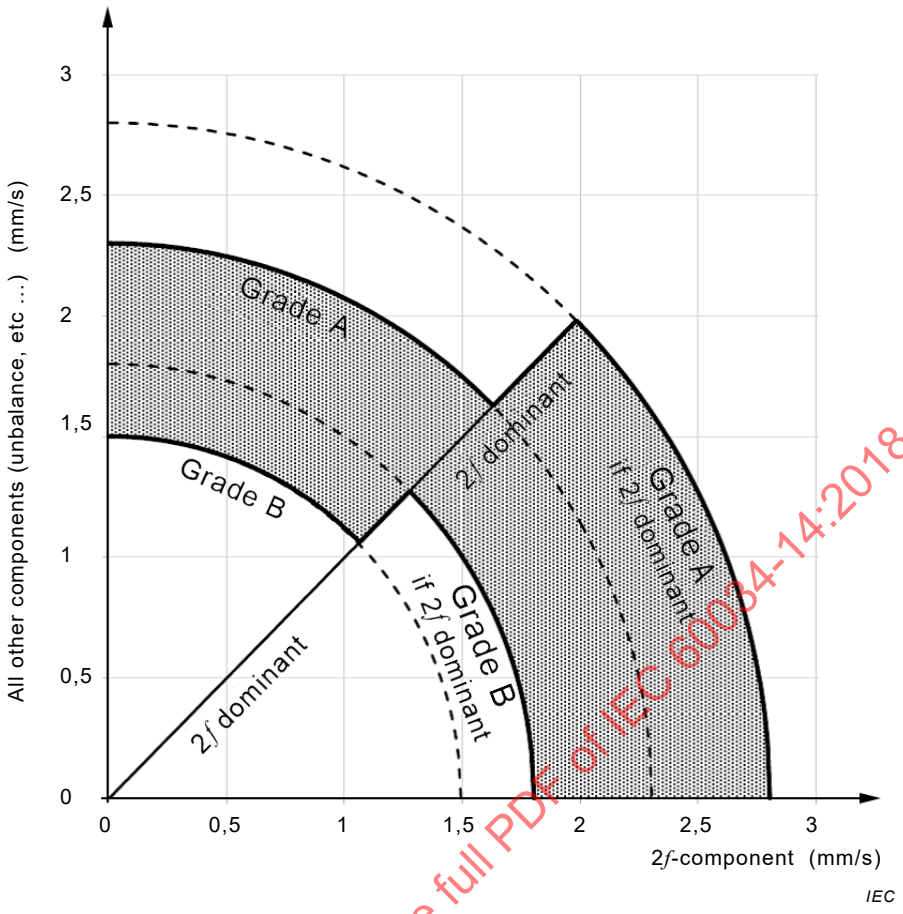


Figure 7 – Vibration limit diagram for shaft heights H > 132 mm

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COMMISSION ÉLECTROTECHNIQUE INTERNATIONALE

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MACHINES ÉLECTRIQUES TOURNANTES –

Partie 14: Vibrations mécaniques de certaines machines de hauteur d'axe supérieure ou égale à 56 mm – Mesurage, évaluation et limites de l'intensité vibratoire

AVANT-PROPOS

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La Norme internationale IEC 60034-14 a été établie par le comité d'études 2 de l'IEC: Machines tournantes.

Cette quatrième édition annule et remplace la troisième édition, parue en 2003 et son amendement 1, paru en 2007. Cette édition constitue une révision technique.

Les modifications techniques majeures par rapport à l'édition précédente sont:

a) Le 6.2 est modifié de manière significative afin de mieux définir «suspension libre».
b) En 6.3, ajout d'une deuxième méthode de montage rigide puisque la première méthode n'est pas toujours réalisable sur le banc d'essai.

c) En 7.1, définition d'option améliorée pour les clavettes d'arbre.

d) À l'Article 8, effort d'harmonisation avec les normes NEMA MG 1, IEEE 841 et API 541, et définition de niveaux réalisables et plus conformes aux meilleures pratiques. Le Tableau 1 est réduit à deux plages de hauteur d'axe.

e) En 8.2, définition simplifiée de «deux fois la fréquence de ligne», et ajout de la Figure 7.

Le texte de cette Norme internationale est issu des documents suivants:

FDIS	Rapport de vote
2/1906/FDIS	2/1914/RVD

Le rapport de vote indiqué dans le tableau ci-dessus donne toute information sur le vote ayant abouti à l'approbation de cette Norme internationale.

Ce document a été rédigé selon les Directives ISO/IEC, Partie 2.

Une liste de toutes les parties de la série IEC 60034, publiées sous le titre général *Machines électriques tournantes*, peut être consultée sur le site web de l'IEC.

NOTE Un tableau de références croisées de toutes les publications du CE 2 de l'IEC est donné sur le tableau de bord du CE 2 sur le site web de l'IEC.

Le comité a décidé que le contenu de ce document ne sera pas modifié avant la date de stabilité indiquée sur le site web de l'IEC sous "http://webstore.iec.ch" dans les données relatives au document recherché. A cette date, le document sera

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MACHINES ÉLECTRIQUES TOURNANTES –

Partie 14: Vibrations mécaniques de certaines machines de hauteur d'axe supérieure ou égale à 56 mm – Mesurage, évaluation et limites de l'intensité vibratoire

1 Domaine d'application

La présente partie de l'IEC 60034 spécifie les procédures d'essai d'acceptation de vibration en usine et les limites de vibration pour certaines machines électriques, dans des conditions spécifiées, sans être couplées à une charge ou à une machine entraînée.

Elle est applicable aux machines à courant continu et aux machines triphasées à courant alternatif de hauteur d'axe supérieure ou égale à 56 mm et de puissance assignée inférieure ou égale à 50 MW, à des vitesses de fonctionnement de 120 min^{-1} jusqu'à et y compris 15 000 min^{-1}.

Le présent document n'est pas applicable aux machines montées *in situ* (sur site), aux moteurs triphasés à collecteurs, aux machines monophasées, aux machines triphasées alimentées en monophasé, aux générateurs hydrauliques verticaux, aux générateurs à turbine de plus de 20 MW et aux machines à paliers magnétiques ou aux machines à enroulement série.

NOTE Pour les machines mesurées *in situ*, se référer aux parties applicables de l'ISO 20816, de l'ISO 10816 et de l'ISO 7919.

2 Références normatives

Les documents suivants cités dans le texte constituent, pour tout ou partie de leur contenu, des exigences du présent document. Pour les références datées, seule l'édition citée s'applique. Pour les références non datées, la dernière édition du document de référence s'applique (y compris les éventuels amendements).

IEC 60034-1, *Machines électriques tournantes – Partie 1: Caractéristiques assignées et caractéristiques de fonctionnement*

IEC 60034-7, *Machines électriques tournantes – Partie 7: Classification des formes de construction et les dispositions de montage (Code IM)*

ISO 2954, *Vibrations mécaniques des machines tournantes ou alternatives – Exigences relatives aux appareils de mesure de l'intensité vibratoire*

ISO 10817-1, *Systèmes de mesure des vibrations des arbres tournants – Partie 1: Captage relatif et captage absolu des vibrations radiales des arbres tournants*

ISO 20816-1, *Vibrations mécaniques – Mesurage et évaluation des vibrations de machines – Partie 1: Lignes directrices générales*

ISO 21940-32, *Vibrations mécaniques – Équilibrage des rotors – Partie 32: Convention relative aux clavettes d'arbres et aux éléments rapportés*

3 Termes et définitions

Pour les besoins du présent document, les termes et définitions donnés dans l'IEC 60034-1 s'appliquent.

L'ISO et l'IEC tiennent à jour des bases de données terminologiques destinées à être utilisées en normalisation, consultables aux adresses suivantes:

- IEC Electropedia: disponible à l'adresse http://www.electropedia.org/
- ISO Online browsing platform: disponible à l'adresse http://www.iso.org/obp

4 Grandeurs mesurées

4.1 Généralités

Les grandeurs mesurées sont le déplacement vibratoire et la vitesse vibratoire aux paliers de la machine et le déplacement vibratoire relatif sur l'arbre à l'intérieur ou près des paliers de la machine.

4.2 Amplitude vibratoire

Le critère à adopter pour l'amplitude vibratoire aux paliers de la machine doit être la valeur efficace large bande du déplacement vibratoire en micromètres ou la vitesse vibratoire en millimètres par seconde dans la plage de fréquences spécifiée à l'Article 5. La valeur maximale déterminée aux positions de mesure et aux grandeurs mesurées spécifiées, conformément à la présente norme, caractérise l'amplitude vibratoire de la machine.

Les moteurs à induction (spécialement du type bipolaire) présentent fréquemment des vitesses vibratoires à deux fois la fréquence de glissement. Dans ces cas, l'amplitude vibratoire doit être déterminée à partir de la relation:

$$x_{\text{r.m.s.}} = \sqrt{\frac{x_{\text{max}}^{\ 2} + x_{\text{min}}^{\ 2}}{2}}$$

où

x_{max} est la valeur efficace maximale du déplacement vibratoire ou de la vitesse vibratoire;

x_{min} est la valeur efficace minimale du déplacement vibratoire ou de la vitesse vibratoire.

La fréquence d'échantillonnage choisie doit être assez grande pour permettre de capturer de manière fiable les valeurs des vibrations maximales et minimales.

NOTE Les grandes machines à induction à courant alternatif fonctionnant à des valeurs de glissement très basses à vide peuvent exiger des temps de mesure de plusieurs minutes à plus de dix minutes pour que les mesurages soient effectués à chaque position de mesure de vibration.

4.3 Vibration relative de l'arbre

Le critère à adopter pour la vibration relative de l'arbre doit être le déplacement vibratoire $S_{\text{p-p}}$ dans la direction de la mesure (selon l'ISO 20816-1).

5 Appareillage de mesure

L'appareillage de mesure doit être capable de mesurer la valeur efficace large bande de vibration en réponse plate sur une plage de fréquences de 10 Hz à 1 000 Hz, conformément aux exigences de l'ISO 2954. Cependant, pour les machines dont la vitesse avoisine ou

descend en dessous de 600 min^{-1}, la limite inférieure de la bande passante ne doit pas être supérieure à 2 Hz.

L'appareillage de mesure pour les mesurages de la vibration relative de l'arbre doit être conforme aux exigences de l'ISO 10817-1.

Les capteurs de vibration multidirectionnels ne doivent pas être utilisés.

NOTE Les capteurs multidirectionnels ne permettent pas d'effectuer un mesurage de vibration approprié dans toutes les directions s'ils sont installés à un seul endroit.

6 Montage de la machine

6.1 Généralités

Les vibrations d'une machine électrique sont étroitement liées au montage de la machine. Pour permettre une évaluation spécifique de l'équilibrage et des vibrations des machines électriques tournantes, il est nécessaire de mesurer les vibrations sur la machine seule, selon des conditions d'essai convenablement définies, de façon à permettre des essais reproductibles et à fournir des mesures comparables.

6.2 Suspension libre

Cet état est obtenu par suspension de la machine à un ressort, ou par montage sur un support élastique (ressorts, caoutchouc, etc.).

La plus haute fréquence d'oscillation naturelle (f_{no}) du système de suspension et de la machine doit être inférieure au tiers de la fréquence f_1 correspondant à la vitesse de la machine en essai, comme défini en 7.3. Basée sur la masse de la machine à l'essai, la rigidité nécessaire du système de suspension en fonction de la vitesse assignée de 600 min^{-1} à 3 600 min^{-1} peut être déterminée à partir de la Figure 1. Pour les vitesses inférieures à 600 min^{-1}, les mesurages en suspension libre ne sont pas pratiques. Pour les vitesses supérieures à 3 600 min^{-1}, il convient que la valeur de déplacement statique Z ne soit pas inférieure à la valeur pour 3 600 min^{-1}.

La courbe de la Figure 1 représente le déplacement élastique minimal pour atteindre l'oscillation naturelle du mouvement de corps rigide vertical nécessaire, qui est habituellement la plus haute des fréquences naturelles de corps rigide. Le déplacement statique Z est exprimé comme suit:

$$Z = \frac{a^2 g}{(2\pi n)^2} \quad , a = \frac{f_1}{f_{no}};\ a \geq 3$$

où

Z est le déplacement en m,

n est la vitesse assignée en unités s^{-1}, et

g est l'accélération due à la pesanteur (9,81 m/s^2).

Lorsque a est réglé sur 3, la courbe de la Figure 1 est générée.

6.3 Montage rigide

6.3.1 Massif

6.3.1.1 Généralités

Lors de l'essai de fonctionnement en atelier de la machine assemblée, les mesurages de vibration doivent être effectués avec la machine correctement calée et solidement fixée à un massif ou à un banc d'essai. Les supports élastiques souples ne sont pas autorisés.

Les fréquences naturelles horizontale et verticale de la configuration d'essai complète ne doivent pas coïncider avec:

a) ± 10 % de la fréquence de rotation de la machine;

b) ± 5 % de deux fois la fréquence de rotation, ou

c) ± 5 % d'une ou deux fois la fréquence électrique de ligne.

Le constructeur peut choisir l'une des deux conditions de montage suivantes.

6.3.1.2 Montage rigide sur massif d'essai

Le massif d'essai est considéré comme satisfaisant lorsque la vitesse vibratoire mesurée selon les directions horizontale et verticale sur les pieds de la machine (ou sur le cadre d'embase près des supports des paliers ou pieds du stator) n'excède pas 30 % de la vitesse maximale, qui est mesurée sur le palier adjacent dans la même direction de la mesure. Le rapport entre les vitesses vibratoires au niveau du pied et du palier est valable pour la composante de fréquence de rotation ou la composante à deux fois la fréquence de ligne (si cette dernière est évaluée).

NOTE 1 La rigidité d'un massif est une grandeur relative. Elle est comparée à la rigidité du système des paliers de la machine. Le rapport entre les vibrations du logement de palier et les vibrations du massif est une grandeur caractéristique pour l'évaluation de la flexibilité du massif.

NOTE 2 Si la machine doit être soutenue par une structure autre qu'un massif d'essai, il peut être nécessaire de réaliser une analyse dynamique du système afin d'apporter les modifications nécessaires à la rigidité dynamique du massif.

6.3.1.3 Montage rigide sur banc d'essai

Cet état est obtenu en montant la machine sur un banc d'essai suffisamment rigide et sans résonance aux fréquences de forçage, voir 6.3.1.1.

NOTE Il s'agit du montage le plus utilisé dans les laboratoires d'essai des constructeurs.

6.3.2 Machines horizontales

La machine en essai doit être boulonnée ou fixée en utilisant toutes les positions de fixation à un massif satisfaisant aux exigences de 6.3.1.2 ou 6.3.1.3.

Il existe des constructions et des montages pour lesquels les conditions de fixation énoncées ci-dessus ne peuvent être satisfaites, comme les machines monopaliers. Dans ces cas, il convient qu'un accord entre le fournisseur et le client soit conclu.

6.3.3 Machines verticales

Les machines verticales doivent être montées sur une plaque d'acier massif de forme rectangulaire ou circulaire, avec un trou au centre du bout d'arbre, la surface étant usinée pour la bride de la machine électrique à l'essai et les trous étant conçus pour les attaches ou les fixations. L'épaisseur de la plaque d'acier doit être au moins trois fois supérieure à l'épaisseur de la bride de la machine, un rapport de cinq est recommandé. La longueur du côté de l'embase relative à son diamètre doit être au moins égale à la hauteur du palier supérieur, L. La Figure 6 est un exemple de IM V1 (voir l'IEC 60034-7).