
**Short-pitch transmission precision
roller and bush chains, attachments
and associated chain sprockets**

*Chaînes de transmission de précision à rouleaux et à douilles, plaques-
attaches et roues dentées correspondantes*

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Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

The procedures used to develop this document and those intended for its further maintenance are described in the ISO/IEC Directives, Part 1. In particular the different approval criteria needed for the different types of ISO documents should be noted. This document was drafted in accordance with the editorial rules of the ISO/IEC Directives, Part 2 (see www.iso.org/directives).

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights. Details of any patent rights identified during the development of the document will be in the Introduction and/or on the ISO list of patent declarations received (see www.iso.org/patents).

Any trade name used in this document is information given for the convenience of users and does not constitute an endorsement.

For an explanation on the meaning of ISO specific terms and expressions related to conformity assessment, as well as information about ISO's adherence to the WTO principles in the Technical Barriers to Trade (TBT) see the following URL: [Foreword - Supplementary information](#)

The committee responsible for this document is ISO/TC 100, *Chains and chain sprockets for power transmission and conveyors*.

This fourth edition cancels and replaces the third edition (ISO 606:2004), which has been technically revised. It also incorporates Technical Corrigendum ISO 606:2004/Cor. 1:2006.

Introduction

The provisions of this revised International Standard have been established by including sizes of chains used by the majority of countries in the world, and by unifying dimensions, strengths and other data which differed in current national standards, while eliminating those for which it was considered a universal usage had not been established.

The whole field of application open to this medium of transmission has been covered by the ranges of chains already established. To achieve this, the sizes of 6,35 mm pitch to 76,2 mm pitch inclusive have been duplicated, on the one hand, by the inclusion of chains derived from standards originating and centred around ANSI, and on the other by chains representing the unification of the principal standards originating in Europe, the two being complementary for the coverage of the widest possible field of application.

The ANSI chain reference numbers (25, 35, 40, 50, etc.) are used world-wide and these numbers have now been introduced into this International Standard in place of the previous ISO reference numbers (04C, 06C, 08A, 10A, etc.) To assist in cross-referencing the ANSI and previous ISO numbers, details are included in [Annex B](#) of this International Standard.

The ANSI heavy series of chains (suffix H) are specified in this International Standard. The ANSI heavy series of chains differs from the ANSI standard series in that thicker plates are used.

The ANSI extra heavy series of chains (suffix HE) are now included. The ANSI extra heavy series are dimensionally as the ANSI heavy series (suffix H) but have a higher minimum ultimate tensile strength.

[Clause 4](#) covers specification details for K and M attachments, and extended pin attachments for use with short-pitch transmission roller and bush chains conforming with this International Standard.

[Clause 5](#), covering chain sprockets, represents the unification of all the relevant national standards in the world and includes, in particular, complete tolerances relating to tooth form.

The inclusion of the dimensions of the chains specified ensures complete interchangeability of any given size and provides interchangeability of individual links of chains.

Short-pitch transmission precision roller and bush chains, attachments and associated chain sprockets

1 Scope

This International Standard specifies the characteristics of short-pitch precision roller and bush chains with associated sprockets suitable for the mechanical transmission of power and allied applications. It covers dimensions, tolerances, length measurement, preloading, minimum tensile strengths and minimum dynamic strength.

Although [Clause 5](#) applies to chain sprockets for cycles and motor cycles, this International Standard is not applicable to their chains, which are covered by ISO 9633 and ISO 10190, respectively.

2 Normative references

The following documents, in whole or in part, are normatively referenced in this document and are indispensable for its application. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

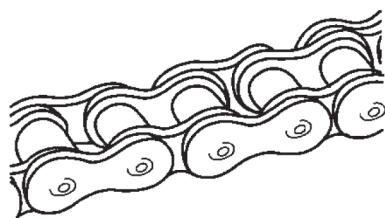
ISO 286-2:2010, *Geometrical product specifications (GPS) — ISO code system for tolerances on linear sizes — Part 2: Tables of standard tolerance classes and limit deviations for holes and shafts*

ISO 15654, *Fatigue test method for transmission precision roller chains*

3 Chains

3.1 Nomenclature of assemblies and components

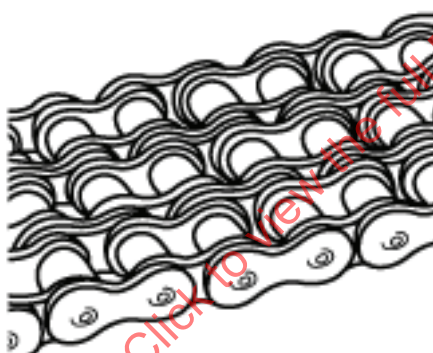
The nomenclature of chain assemblies and their component parts is shown in [Figures 1](#) and [2](#) (which do not define the actual form of the chain plates).



a) Simplex chain

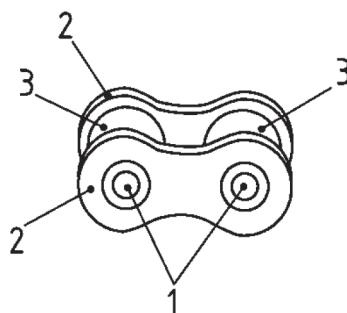


b) Duplex chain



c) Triplex chain

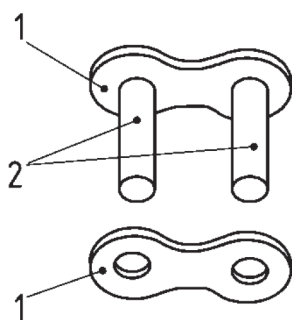
Figure 1 — Types of roller chain assembly



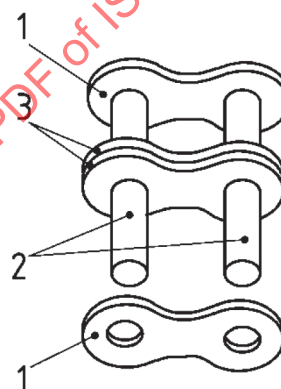
a) Inner link

Key for a)

- 1 bush
- 2 inner plate
- 3 roller



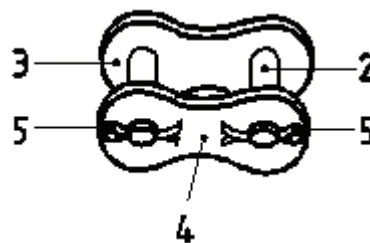
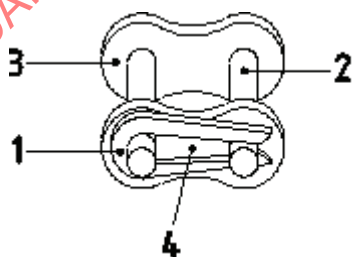
Simplex outer link



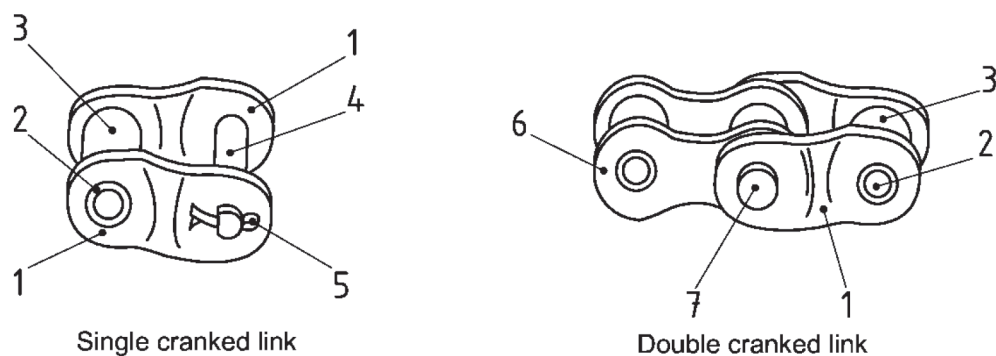
Duplex outer link

b) Outer links for riveting**Key for b)**

- 1 outer plate
- 2 bearing pins
- 3 intermediate plate(s)

**c) Detachable connecting links****Key for c)**

- 1 spring clip fastener
- 2 fixed connecting pin
- 3 outer plate
- 4 detachable plate
- 5 cotter pin fastener



d) Cranked links

Key for d)

1	cranked plate	5	cotter pin fastener
2	bush	6	inner plate
3	roller	7	bearing pin, riveted
4	detachable connecting pin		

NOTE 1 The plate dimensions are specified in [Tables 1](#) and [2](#).

NOTE 2 Fasteners can be of various designs. Drawings indicate examples.

Figure 2 — Types of link

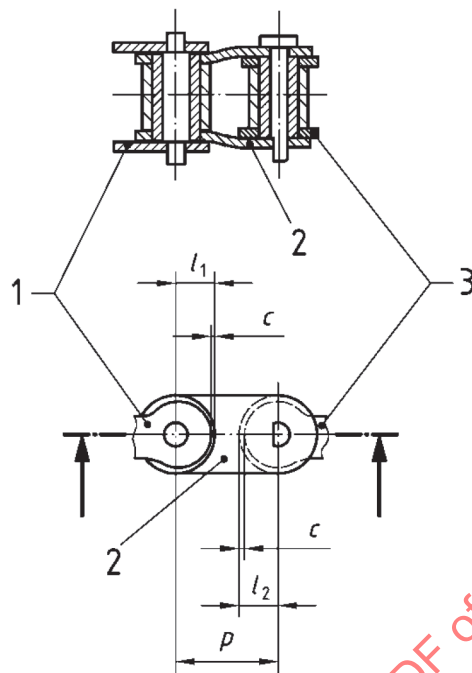
3.2 Designation

Chains are designated by the standard ISO chain number given in [Tables 1](#) and [2](#). The ISO chain numbers in [Table 1](#) are supplemented by a hyphenated suffix 1 for simplex chain, 2 for duplex chain and 3 for triplex chain, for example, 16B-1, 16B-2, 16B-3, 80-1, 80-2, 80-3. Chains 081, 083, 084 and 41 do not follow this procedure since they are normally available in simplex form only.

The chains designated in [Table 2](#) are the ANSI heavy and extra heavy series, which are also supplemented by a hyphenated suffix 1 for simplex chain, 2 for duplex chain and 3 for triplex chain, for example, 80H-1, 80H-2, 80H-3, 80HE-1, 80HE-2, 80HE-3.

3.3 Dimensions

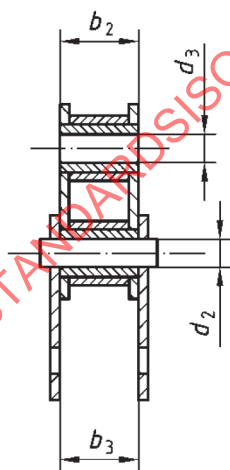
Chains shall conform to the dimensions shown in [Figure 3](#) and given in [Tables 1](#) and [2](#). Maximum and minimum dimensions are specified to ensure interchangeability of links produced by different makers of chain. They represent limits for interchangeability, but are not the manufacturing tolerances.



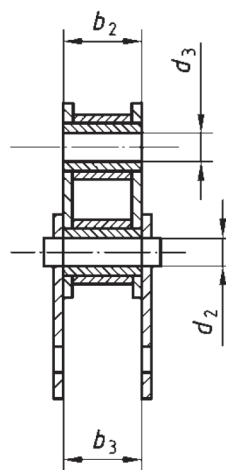
a) Cranked link

Key for a)

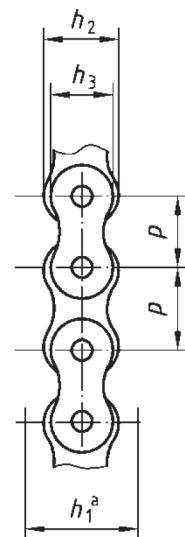
- c clearance between cranked link plates and straight plates available during articulation
- p pitch
- 1 outer plate
- 2 cranked plate
- 3 inner plate



Plain pin



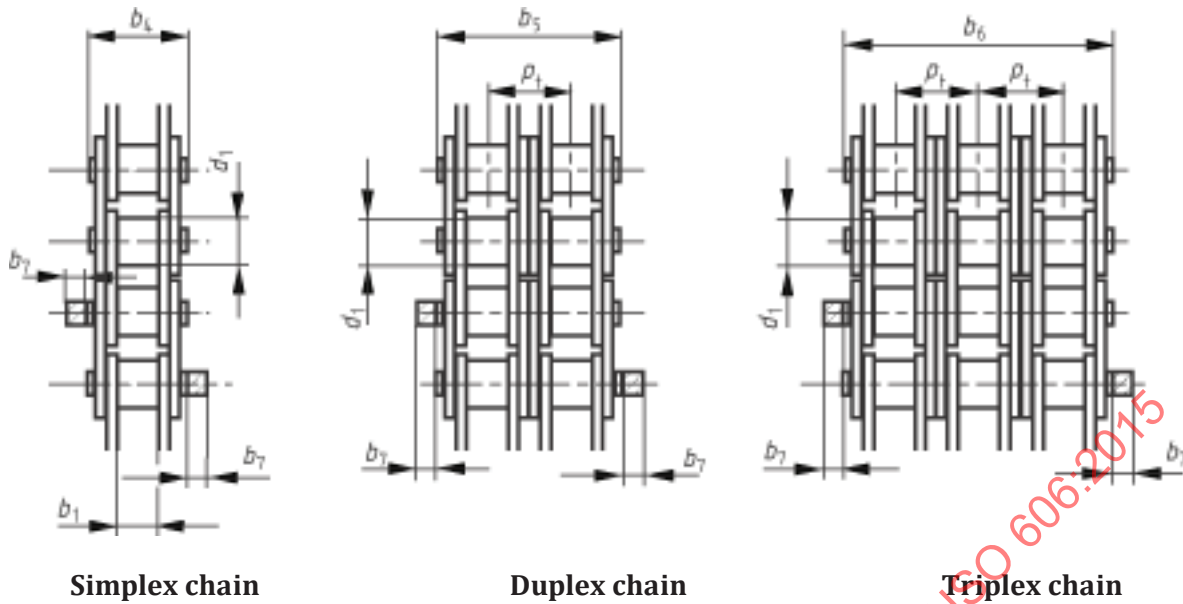
Shouldered pin



b) Sections through chain

Key for b)

- ^a Chain path depth is the minimum depth of channel through which the assembled chain will pass



c) Types of chain

NOTE For the symbols, see [Table 1](#).

Figure 3 — Chains

The overall width of a simplex, duplex or triplex chain with a joint fastener is given by

a) for riveted pin end chains if the fastener is on one side only:

$$(b_4 + b_7) \text{ or } (b_5 + b_7) \text{ or } (b_6 + b_7);$$

b) for riveted pin end chains if the fastener is on two sides:

$$[b_4 + (2b_7)] \text{ or } [b_5 + (2b_7)] \text{ or } [b_6 + (2b_7)];$$

c) for headed pin end chains if the fastener is on one side only:

$$[b_4 + (1,6b_7)] \text{ or } [b_5 + (1,6b_7)] \text{ or } [b_6 + (1,6b_7)];$$

d) for headed pin end chains if the fastener is on two sides:

$$[b_4 + (3,2b_7)] \text{ or } [b_5 + (3,2b_7)] \text{ or } [b_6 + (3,2b_7)].$$

The overall width of chains wider than triplex is given by

$$b_4 + [p_t \times (\text{number of strands in chain} - 1)].$$

3.4 Performance requirements

3.4.1 General

WARNING — The test requirements are not to be taken as working loads. These loads could be selected, indirectly, using ISO 10823. The test results shall be invalid if the chain has previously been in service or stressed in any way (other than by preloading in accordance with [3.4.3](#)).

The tests given in [3.4.2](#) to [3.4.5](#) shall only be performed on unused, undamaged chains to determine whether the subject chain complies with the minimum requirements specified in [Tables 1](#) and [2](#).

3.4.2 Tensile testing

3.4.2.1 The minimum tensile strength is that value which shall be exceeded when a tensile force is applied to a sample tested to destruction in accordance with [3.4.2.2](#)

NOTE This minimum tensile strength is not a working load, but is intended primarily as a comparative figure between chains of various constructions.

3.4.2.2 Apply a tensile force slowly to the ends of a chain length containing at least five free pitches by means of fixtures permitting free movement on both sides of the chain centreline, in the normal plane of articulation. [Annex E](#) (informative) describes methods to consider using in order to avoid an excessive increase in the rate of stress being applied to the chain during the tensile test.

Failure shall be considered to have occurred at the first point where increasing extension is no longer accompanied by increasing force, i.e. the summit of the force/extension diagram. The force at this point shall exceed the minimum tensile strength stated in [Tables 1](#) and [2](#).

Tests in which failures occur adjacent to the shackles shall be disregarded.

3.4.2.3 The tensile test shall be considered as a destructive test. Even though a chain might not visibly fail when subjected to a force equivalent to the minimum tensile strength, it will have been stressed beyond the yield point and will be unfit for service.

3.4.2.4 These requirements do not apply to cranked links, connecting links or chains with attachments, as their tensile strength could be reduced.

3.4.3 Preloading

Chains manufactured in conformance with this International Standard shall be preloaded by applying a minimum tensile force equivalent to 30 % of the minimum tensile strength given in [Tables 1](#) and [2](#).

3.4.4 Length validation

Measurement of chains shall take place after preloading but before lubrication.

The standard length for measurement shall be a minimum of

- a) 610 mm for ISO chain numbers 25 to 12B and 081 to 41 inclusive, or
- b) 1 220 mm for ISO chain numbers 80 to 72B inclusive.

The chain shall be supported throughout its entire length and the measuring force specified in [Tables 1](#) and [2](#) shall be applied.

The measured length shall be the nominal length $^{+0,15}_0\%$, except for chains with attachments when it shall be the nominal length $^{+0,30}_0\%$.

The length accuracy of chains which have to work in parallel can be matched within closer tolerances.

3.4.5 Dynamic testing

Chains in conformance with this International Standard shall survive a conformance test, as specified in ISO 15654, using the dynamic strength values given in [Tables 1](#) or [2](#) for the particular chain. These requirements do not apply to cranked links, connecting links or chains with attachments, as their dynamic strength could be reduced. The methods used for calculating the minimum dynamic strength are given in [Annex C](#). The method for determining the maximum test force for the conformance test is given in [Annex D](#). The informative [Annex F](#) describes two methods used to approximate the minimum dynamic test values for multiplex chains.

3.5 Marking

The chain shall be marked with the manufacturer's name or trademark. The chain number quoted in [Tables 1](#) or [2](#) should be marked on the chain, with the exception of chains in the ANSI extra heavy series (suffix HE) where the manufacturer can mark the chains with their own designation.

3.6 Cranked links

Cranked links should not be used with the heavy and extra heavy series chains or on chains which are intended for highly stressed applications. Where a cranked link is used a reduction in performance will occur.

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Table 1 — (continued)

ISO Chain number ^a	Pitch	Maximum roller diameter	Minimum width between inner plates	Maximum bearing pin body diameter	Minimum bush bore	Minimum chain path depth	Maximum inner plate depth	Maximum outer or intermediate plate depth	Minimum cranked link dimensions ^b			Transverse pitch	Maximum width over inner link	Minimum width between outer plates	Maximum width over bearing pins			Maximum additional width for joint fastener ^c	Measuring force				Minimum tensile strength				Minimum dynamic strength ^{d, e, f}		
									l ₁	l ₂	c				Simplex	Duplex	Triplex		Simplex	Duplex	Triplex	Simplex	Duplex	Triplex					
mm																													
28B	44,45	27,94	30,99	15,90	15,95	37,46	37,08	37,08	19,51	19,51	0,20	59,56	46,58	46,71	65,1	124,7	184,3	7,4	1 510	3 020	4 540	200,0	360,0	530,0				N	27 100
160	50,80	28,58	31,55	14,29	14,31	48,74	48,26	41,68	21,04	24,33	0,20	58,55	45,21	45,26	65,5	124,2	182,9	7,9	2 000	4 000	6 010	223,0	446,0	669,0					34 800
32B	50,80	29,21	30,99	17,81	17,86	42,72	42,29	42,29	22,20	22,20	0,20	58,55	45,57	45,70	67,4	126,0	184,5	7,9	2 000	4 000	6 010	250,0	450,0	670,0					29 900
180	57,15	35,71	35,48	17,46	17,49	54,86	54,30	46,86	23,65	27,36	0,20	65,84	50,85	50,90	73,9	140,0	206,0	9,1	2 670	5 340	8 010	281,0	562,0	843,0					44 500
200	63,50	39,68	37,85	19,85	19,87	60,93	60,33	52,07	26,24	30,36	0,20	71,55	54,68	54,94	80,3	151,9	223,5	10,2	3 110	6 230	9 340	347,0	694,0	1 041,0					53 600
40B	63,50	39,37	38,10	22,89	22,94	53,49	52,96	52,96	27,76	27,76	0,20	72,29	55,75	55,88	82,6	154,9	227,2	10,2	3 110	6 230	9 340	355,0	710,0	1 065,0					41 800
240	76,20	47,63	47,35	23,81	23,84	73,13	72,39	62,49	31,45	36,40	0,20	87,83	67,81	67,87	95,5	183,4	271,3	10,5	4 450	8 900	13 340	500,0	1 000,0	1 500,0					73 100
48B	76,20	48,26	45,72	29,24	29,29	64,52	63,88	63,88	33,45	33,45	0,20	91,21	70,56	70,69	99,1	190,4	281,6	10,5	4 450	8 900	13 340	560,0	1 000,0	1 500,0					63 600
56B	88,90	53,98	53,34	34,32	34,37	78,64	77,85	77,85	40,61	40,61	0,20	106,60	81,33	81,46	114,6	221,2	327,8	11,7	6 090	12 190	20 000	850,0	1 600,0	2 240,0					88 900
64B	101,60	63,50	60,96	39,40	39,45	91,08	90,17	90,17	47,07	47,07	0,20	119,89	92,02	92,15	136,9	250,8	370,7	13,0	7 960	15 920	27 000	1 120,0	2 000,0	3 000,0					106 900
72B	114,30	72,39	68,58	44,48	44,53	104,67	103,63	103,63	53,37	53,37	0,20	136,27	103,81	103,94	147,4	283,7	420,0	14,3	10 100	20 190	33 500	1 400,0	2 500,0	3 750,0					132 700

^a For details of heavy and extra heavy series chains, see Table 2.

^b Cranked links are not recommended for use of highly stressed applications.

^c The actual dimensions will depend on the type of fastener used, they should not exceed the dimensions given, details of which should be obtained by the purchaser from the manufacturer.

^d These dynamic strength values do not apply to cranked links or connecting links or chains with attachments.

^e Whilst dynamic test values for duplex and triplex chains should not be proportioned from the simplex test value, Appendix F details methods of calculation for when an approximation is required.

^f Dynamic strength values are based on test specimens each of 5 free pitches except for 180, 200, 40B, 240, 48B, 56B, 64B and 72B chains which are based on test specimens each of 3 free pitches. See Annex C for method of calculation.

^g Bush diameter.

^a For details of heavy and extra heavy series chains, see Table 2.

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^g Bush diameter.

Table 2 — Principal dimensions, measuring forces, tensile strengths and dynamic strength values of ANSI heavy and extra heavy series chains

ISO Chain number ^a	Pitch	Maximum roller diameter	Minimum width between inner plates	Maximum bearing pin body diameter	Minimum chain pin depth	Maximum inner plate depth	Maximum outer or intermediate plate depth	Minimum cranked link dimensions ^b			Transverse pitch	Maximum width over inner link	Minimum width between outer plates	Maximum width over bearing pins			Maximum additional width ^c for joint fastener	Measuring force			Minimum tensile strength			Minimum dynamic strength ^{d, e, f}																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																										
								d ₁	b ₁	d ₂				d ₃	h ₁	h ₂		h ₃	l ₁	l ₂	c	p	b ₂		b ₃	Chain			Chain			F _u	F _d																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																	
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^a For details of standard series chains, see Table 1.

^b Cranked links are not recommended for use on highly stressed applications.

^c The actual dimensions will depend on the type of fastener used, but they should not exceed the dimensions given, details of which should be obtained by the purchaser from the manufacturer.

^d These dynamic strength values do not apply to cranked links or connecting links or chains with attachments.

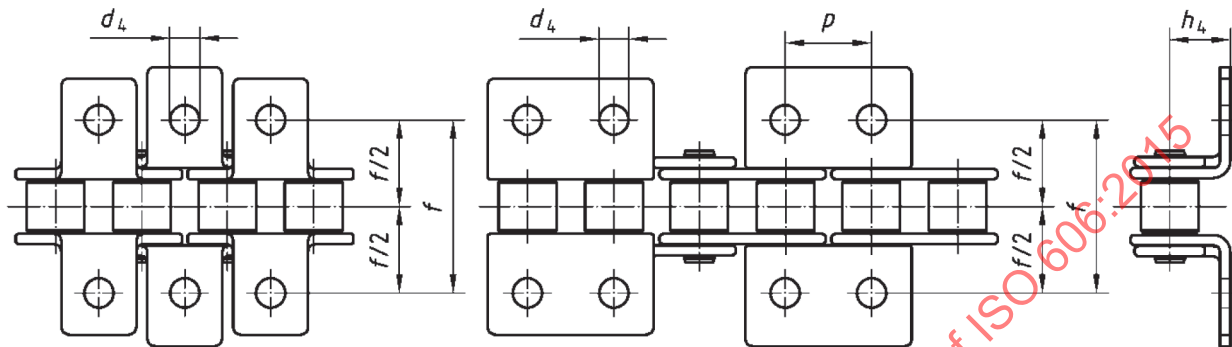
^e Whilst dynamic test values for duplex and triplex chains should not be proportioned from the simplex test value, Appendix F details methods of calculation for when an approximation is required.

^f Dynamic strength values are based on test specimens each of 5 free pitches except for 180H, 180HE, 200H, 200HE, 240H and 240HE chains which are based on test specimens each of 3 free pitches. See Annex C for method of calculation.

4 Attachments

4.1 Nomenclature

The nomenclature for chain attachments is given in [Figures 4, 5, 6 and 7](#), and in [Tables 1, 3, 4 and 5](#).



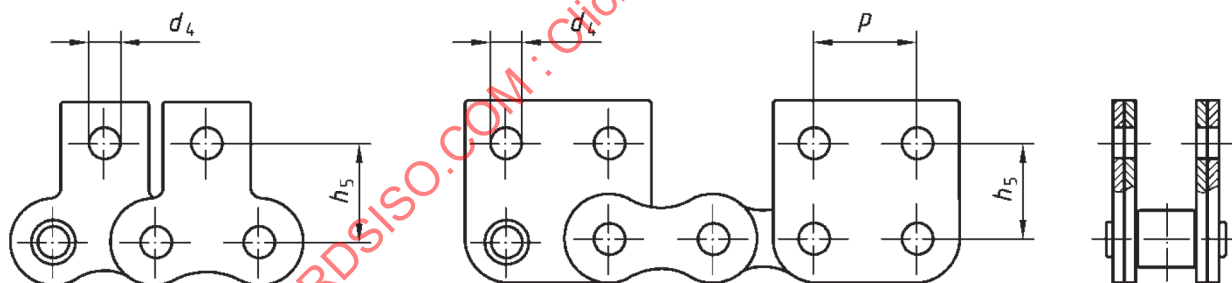
NOTE 1 For d_4 , h_4 and f , see [Table 3](#); for p , see [Table 1](#).

NOTE 2 K attachment plates can be positioned on either outer or inner links.

NOTE 3 K1 plates could be identical to K2 plates except that they have one hole located centrally.

NOTE 4 The assembly of K2 plates on adjacent links is not possible.

Figure 4 — K attachment plates



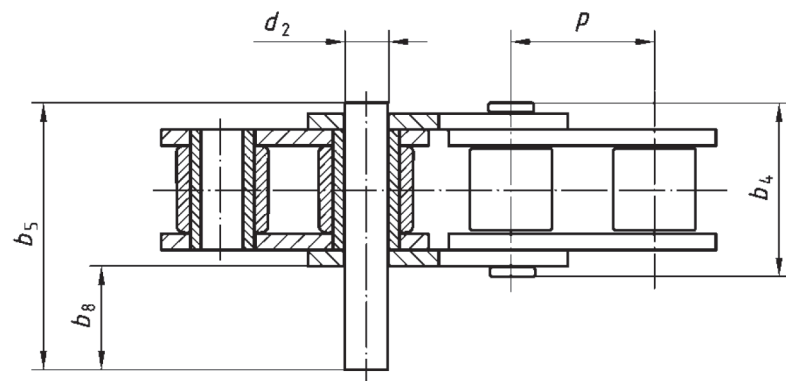
NOTE 1 For d_4 and h_5 , see [Table 4](#); for p , see [Table 1](#).

NOTE 2 M attachment plates can be positioned on either outer or inner links.

NOTE 3 M1 plates could be identical to M2 plates except that they have one hole located centrally.

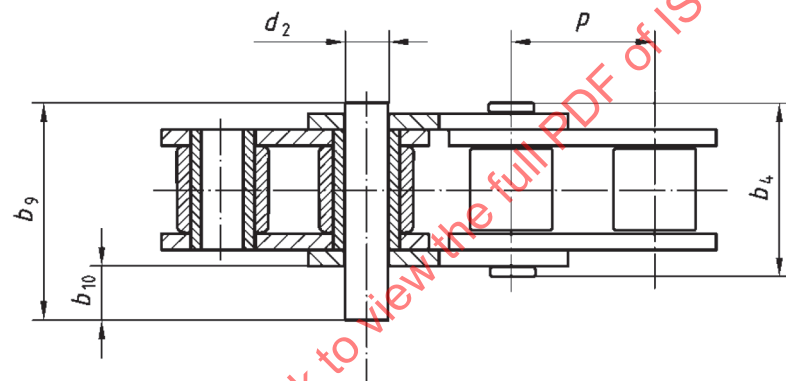
NOTE 4 The assembly of M2 plates on adjacent links is not recommended.

Figure 5 — M attachment plates



NOTE For b_4 and p , see [Table 1](#); for b_5 , b_8 and d_2 , see [Table 5](#).

Figure 6 — Extended bearing pins (based on duplex pin) — Type X



NOTE For b_4 and p , see [Table 1](#); for b_9 , b_{10} and d_2 , see [Table 5](#).

Figure 7 — Extended bearing pins (commonly used in "ANSI" series) — Type Y

4.2 General

Except when otherwise stated, the characteristics, dimensions and tests for the chain with attachments shall be in accordance with [Clause 3](#).

4.3 Designation

Three types of attachment are given, with the common dimensional basis as given in [Tables 3, 4](#) and [5](#). Their designation and distinguishing features are as follows:

- a) K attachments, as shown in [Figure 4](#):
 - 1) K1, with one attachment hole centrally located in each platform;
 - 2) K2, with two attachment holes longitudinally located;
- b) M attachments, as shown in [Figure 5](#):
 - 1) M1, with one attachment hole centrally located in the plate;

- 2) M2, with two attachment holes longitudinally located;
- c) Extended pin: with the bearing pin extended on one side of the chain as shown in [Figures 6](#) and [7](#). Alternative pin extensions are shown, one based on the use of the duplex pin (see [Figure 6](#)) and the other based on those extended pins commonly used in “ANSI” series chains (see [Figure 7](#)).

4.4 Dimensions

Attachments shall conform to the dimensions given in [Tables 3, 4](#) and [5](#).

4.5 Manufacture

The actual form of the attachment plates is left to the discretion of the manufacturer. K attachment plates are normally bent from M attachment plates.

The length of the attachment plate is also left to the discretion of the manufacturer, but should be sufficient to accommodate the two attachment holes longitudinally in the case of Type K2 and not interfere with the working of the adjoining links. A common length could be adopted for both Type K1 and K2.

4.6 Marking

It is not a requirement that K and M attachment plates be marked.

The marking of the extended pin chain shall be the same as that which would be shown on a chain with no attachments (see [3.5](#)).

Table 3 — Attachment plate K — Dimensions (see [Figure 4](#))

ISO chain number	Platform height	Minimum hole diameter	Traverse distance between hole centres
	h_4 mm	d_4 mm	f mm
35	6,4	2,6	19,0
40	7,9	3,3	25,4
08B	8,9	4,3	25,4
50	10,3	5,1	31,8
10B	10,3	5,3	31,8
60	11,9	5,1	38,1
12B	13,5	6,4	38,1
80	15,9	6,6	50,8
16B	15,9	6,4	50,8
100	19,8	8,2	63,5
20B	19,8	8,4	63,5
120	23,0	9,8	76,2
24B	26,7	10,5	76,2
140	28,6	11,4	88,9
28B	28,6	13,1	88,9
160	31,8	13,1	101,6
32B	31,8	13,1	101,6
200	42,9	16,3	127,0

Table 4 — Attachment plate M — Dimensions (see Figure 5)

ISO chain number	Height from chain centre line	Minimum diameter of holes
	h_5 mm	d_4 mm
35	9,5	2,6
40	12,7	3,3
08B	13,0	4,3
50	15,9	5,1
10B	16,5	5,3
60	18,3	5,1
12B	21,0	6,4
80	24,6	6,6
16B	23,0	6,4
100	31,8	8,2
20B	30,5	8,4
120	36,5	9,8
24B	36,0	10,5
140	44,4	11,4
160	50,8	13,1
200	63,5	16,3

Table 5 — Extended pin dimensions (see [Figures 6](#) and [7](#))

Dimensions in millimetres

ISO chain number	Pin extension		Pin extension ^a		Pin diameter
	Type "X"		Type "Y"		Types "X" and "Y"
	b_8 max.	b_5 max.	b_{10} max.	b_9 max.	d_2 max.
05B	7,1	14,3	—	—	2,31
35	12,3	23,4	10,2	21,9	3,60
06B	12,2	23,8	—	—	3,28
40	16,5	32,3	10,2	26,3	3,98
08B	15,5	31,0	—	—	4,45
50	20,6	39,9	12,7	32,6	5,09
10B	18,5	36,2	—	—	5,08
60	25,7	49,8	15,2	40,0	5,96
12B	21,5	42,2	—	—	5,72
80	32,2	62,7	20,3	51,7	7,94
16B	34,5	68,0	—	—	8,28
100	39,1	77,0	25,4	63,8	9,54
20B	39,4	79,7	—	—	10,19
120	48,9	96,3	30,5	78,6	11,11
24B	51,4	101,8	—	—	14,63
140	—	—	35,6	87,5	12,71
160	—	—	40,6	102,6	14,29

^a The pin extensions of Type "Y" are given as alternatives, as they are commonly used in "ANSI" series chains.

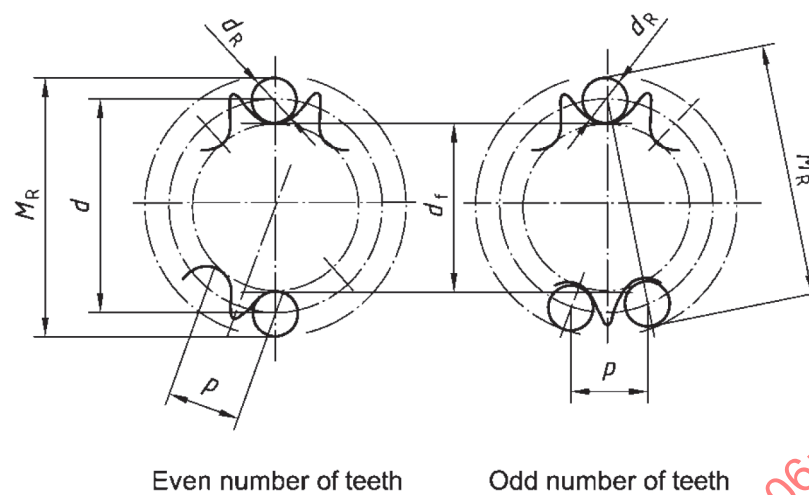
5 Chain sprockets

5.1 General

This clause gives specifications for chain sprockets for use with short-pitch transmission precision roller and bush chains conforming to [Clause 3](#) and specifies general criteria for ensuring correct meshing, operation and transmission of load when used under normal operating conditions.

5.2 Nomenclature

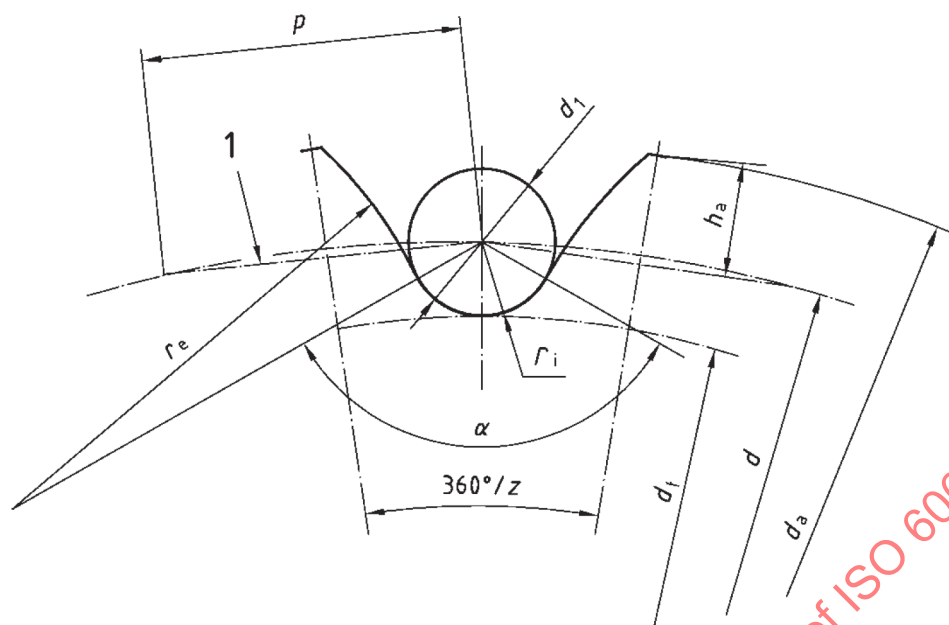
The nomenclature for chain sprockets is shown in [Figures 8, 9](#) and [10](#).

**Key**

- p chordal pitch, equal to chain pitch
 d_R measuring-pin diameter
 d pitch-circle diameter
 d_f root diameter
 M_R measurement over pins

NOTE This nomenclature is valid for both roller and bush chains.

Figure 8 — Chain sprocket diametral dimensions

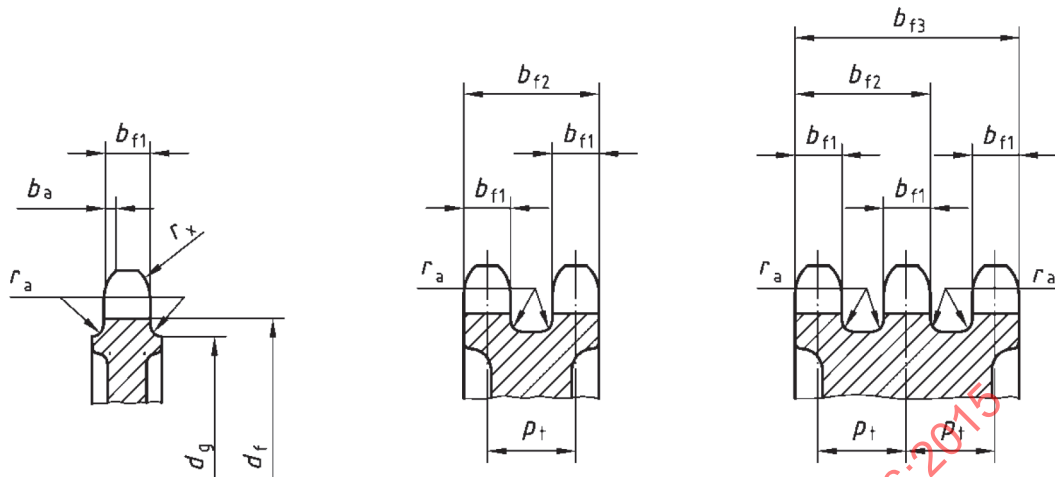


Key

- 1 pitch polygon
- p chordal pitch, equal to chain pitch
- d pitch-circle diameter
- d_1 maximum roller diameter
- r_i roller-seating radius
- α roller-seating angle

- r_e tooth-flank radius
- h_a height of tooth above pitch polygon
- d_a tip diameter
- d_f root diameter
- z number of teeth

Figure 9 — Tooth gap forms

**Key**

b_a	tooth side relief	d_g	absolute maximum shroud diameter
b_{f1}	tooth width	p_t	strand transverse pitch
b_{f2}, b_{f3}	width over teeth	r_a	shroud fillet radius
d_f	sprocket root diameter	r_x	tooth side radius

NOTE For a sprocket rim in the axial plane sectioned through the centre of the tooth gap.

Figure 10 — Sprocket rim profiles

5.3 Diametral dimensions of sprocket rim

5.3.1 Nomenclature

See [Figure 8](#).

5.3.2 Dimensions

5.3.2.1 Pitch-circle diameter, d

The chain sprocket pitch-circle diameter, d , is given by

$$d = \frac{p}{\sin \frac{180^\circ}{z}}$$

[Annex A](#) gives the pitch-circle diameter for unit pitch as a function of the number of teeth.

5.3.2.2 Measuring-pin diameter, d_R

The chain sprocket measuring-pin diameter d_R is given by

$$d_R = d_1$$

(see [Figure 9](#))

with a tolerance of $^{+0,01}_0$ mm.

5.3.2.3 Root diameter, d_f

The chain sprocket root diameter d_f is given by

$$d_f = d - d_1$$

with a tolerance in accordance with [Table 6](#).

Table 6 — Root diameter tolerances

Dimensions in millimetres

Root diameter d_f	Tolerance
$d_f \leq 127$	$\begin{matrix} 0 \\ -0,25 \end{matrix}$
$127 < d_f \leq 250$	$\begin{matrix} 0 \\ -0,3 \end{matrix}$
$d_f > 250$	h11 ^a
^a See ISO 286-2.	

5.3.2.4 Measurement over pins

For an even number of teeth, the measurement over pins is given by

$$M_R = d + d_{R, \min}$$

For an odd number of teeth, the measurement over pins is given by

$$M_R = d \cos \frac{90^\circ}{z} + d_{R, \min}$$

The measurement over pins of sprockets with an even number of teeth shall be carried out over pins inserted in opposite tooth gaps.

The measurement over pins of sprockets with an odd number of teeth shall be carried out over pins in the tooth gaps most nearly opposite.

The limits of tolerance for measurement over pins are identical to those for the corresponding root diameters.

5.4 Sprocket tooth gap forms

5.4.1 Nomenclature

See [Figure 9](#).

5.4.2 Dimensions

5.4.2.1 General

The limits of the tooth gap form are determined by the minimum and maximum tooth gap forms. The actual tooth gap form, which is provided by cutting or an equivalent method, shall have tooth flanks of a form lying between the minimum and maximum flank radii and blending smoothly with the roller seating curve subtending the respective angles.

5.4.2.2 Minimum form

The corresponding values for r_e , r_i and α are given by

$$r_{e,\max} = 0,12d_1 (z + 2)$$

$$r_{i,\min} = 0,505d_1$$

$$\alpha_{\max} = 140^\circ - \frac{90^\circ}{z}$$

5.4.2.3 Maximum form

The corresponding values for r_e , r_i and α are given by

$$r_{e,\min} = 0,008d_1 (z^2 + 180)$$

$$r_{i,\max} = 0,505d_1 + 0,069\sqrt[3]{d_1}$$

$$\alpha_{\min} = 120^\circ - \frac{90^\circ}{z}$$

5.5 Tooth heights and tip diameters

5.5.1 Nomenclature

See [Figure 9](#).

5.5.2 Dimensions

The maximum and minimum values of the tip diameter d_a are given by

$$d_{a,\max} = d + 1,25p - d_1$$

$$d_{a,\min} = d + p \left(1 - \frac{1,6}{z} \right) - d_1$$

NOTE $d_{a,\min}$ and $d_{a,\max}$ can be applied arbitrarily both to the minimum and maximum gap forms, subject to the limitations imposed on the maximum diameter by the cutter.

To facilitate the construction of the tooth gap form to a large scale, the tooth height above the pitch polygon can be obtained from the following formulae:

$$h_{a,\max} = 0,625p - 0,5d_1 + \frac{0,8p}{z}$$

$$h_{a,\min} = 0,5 (p - d_1)$$

NOTE $h_{a,\max}$ is related to $d_{a,\max}$ and $h_{a,\min}$ to $d_{a,\min}$.

5.6 Sprocket rim profiles

5.6.1 Nomenclature

See [Figure 10](#).

5.6.2 Dimensions

5.6.2.1 Tooth width

Tooth width dimensions are given by the following:

a) for $P \leq 12,7$ mm:

- 1) $b_{f1} = 0,93b_1 : h14^{1)}$ for simplex chain sprockets;
- 2) $b_{f1} = 0,91b_1 : h14$ for duplex and triplex chain sprockets;
- 3) $b_{f1} = 0,88b_1 : h14$ for quadruplex chain wheels and above.

b) for $P > 12,7$ mm:

- 1) $b_{f1} = 0,95b_1 : h14$ for simplex chain sprockets;
- 2) $b_{f1} = 0,93b_1 : h14$ for duplex and triplex chain sprockets;
- 3) The formulae given in a) for quadruplex chains and above may be used by agreement between user and manufacturer.

5.6.2.2 Other dimensions

For all chains: b_{f2} and $b_{f3} = (\text{number of strands} - 1) \times p_t + b_{f1}$ (tolerance $h14^{2)}$ on b_{f1})

For all chains: $r_{x,nom} = p$

For chain numbers 081, 083, 084 and 41: $b_{a,nom} = 0,06p$

For all other chains: $b_{a,nom} = 0,13p$

For chain numbers 25 and 35: $d_g = p \cot \frac{180^\circ}{z} - 1,05h_2 - 1,00 - 2r_a$

For all other chains: $d_g = p \cot \frac{180^\circ}{z} - 1,04h_2 - (0,76 \text{ mm})$

5.7 Radial run-out

Radial run-out between the bore and root diameter shall not exceed a total indicator reading of greater than the larger of the two following values:

$(0,0008 d_f + 0,08) \text{ mm}$, or

0,15 mm,

up to a maximum of 0,76 mm.

1) See ISO 286-2.

2) See ISO 286-2.

5.8 Axial run-out (wobble)

Axial run-out, measured with reference to the bore and the flat part of the side face of the teeth, shall not exceed a total indicator reading of

$$(0,000\,9\,d_f + 0,08)\text{ mm},$$

up to a maximum of 1,14 mm.

For fabricated (welded) sprockets, 0,25 mm is acceptable if the above formula gives smaller values.

5.9 Pitch accuracy of sprocket teeth

Pitch accuracy of sprocket teeth is important and chain manufacturers should be consulted for details.

5.10 Number of teeth

This International Standard primarily applies to a number of teeth from 9 to 150 inclusive.

The preferred numbers of teeth are 17, 19, 21, 23, 25, 38, 57, 76, 95 and 114.

5.11 Bore tolerance

Unless otherwise agreed between manufacturer and purchaser, bore tolerance shall be H8³⁾.

5.12 Marking

Sprockets shall be marked with the following:

- a) manufacturer's name or trademark;
- b) number of teeth;
- c) chain designation (ISO chain number and/or manufacturer's equivalent).

3) See ISO 286-2.

Annex A (normative)

Pitch circle diameters

[Table A.1](#) gives the correct pitch circle diameters for sprockets to suit a chain of unit pitch. The pitch circle diameters for sprockets to suit a chain of any other pitch are directly proportional to the pitch of the chain.

Table A.1 — Pitch circle diameters

Number of teeth <i>z</i>	Pitch circle diameter, <i>d</i> , for unit pitch ^a mm	Number of teeth <i>z</i>	Pitch circle diameter, <i>d</i> , for unit pitch ^a mm	Number of teeth <i>z</i>	Pitch circle diameter, <i>d</i> , for unit pitch ^a mm
9	2,923 8	32	10,202 3	55	17,516 6
10	3,236 1	33	10,520 1	56	17,834 7
11	3,549 4	34	10,838 0	57	18,152 9
12	3,863 7	35	11,155 8	58	18,471 0
13	4,178 6	36	11,473 7	59	18,789 2
14	4,494 0	37	11,791 6	60	19,107 3
15	4,809 7	38	12,109 6	61	19,425 5
16	5,125 8	39	12,427 5	62	19,743 7
17	5,442 2	40	12,745 5	63	20,061 9
18	5,758 8	41	13,063 5	64	20,380 0
19	6,075 5	42	13,381 5	65	20,698 2
20	6,392 5	43	13,699 5	66	21,016 4
21	6,709 5	44	14,017 6	67	21,334 6
22	7,026 6	45	14,335 6	68	21,652 8
23	7,343 9	46	14,653 7	69	21,971 0
24	7,661 3	47	14,971 7	70	22,289 2
25	7,978 7	48	15,289 8	71	22,607 4
26	8,296 2	49	15,607 9	72	22,925 6
27	8,613 8	50	15,926 0	73	23,243 8
28	8,931 4	51	16,244 1	74	23,562 0
29	9,249 1	52	16,562 2	75	23,880 2
30	9,566 8	53	16,880 3	76	24,198 5
31	9,884 5	54	17,198 4	77	24,516 7

^a This is sometimes referred to as “unit pitch circle diameter”.

Table A.1 (continued)

Number of teeth <i>z</i>	Pitch circle diameter, <i>d</i> , for unit pitch ^a mm	Number of teeth <i>z</i>	Pitch circle diameter, <i>d</i> , for unit pitch ^a mm	Number of teeth <i>z</i>	Pitch circle diameter, <i>d</i> , for unit pitch ^a mm
78	24,334 9	105	33,427 5	132	42,020 9
79	25,153 1	106	33,745 8	133	42,339 1
80	25,471 3	107	34,064 0	134	42,657 4
81	25,789 6	108	34,382 3	135	42,975 7
82	26,107 8	109	34,700 6	136	43,294 0
83	26,426 0	110	35,018 8	137	43,612 3
84	26,744 3	111	35,337 1	138	43,930 6
85	27,062 5	112	35,655 4	139	44,248 8
86	27,380 7	113	35,973 7	140	44,567 1
87	27,699 0	114	36,291 9	141	44,885 4
88	28,017 2	115	36,610 2	142	45,203 7
89	28,335 5	116	36,928 5	143	45,522 0
90	28,653 7	117	37,246 7	144	45,840 3
91	28,971 9	118	37,565 0	145	46,158 5
92	29,290 2	119	37,883 3	146	46,476 8
93	29,608 4	120	38,201 6	147	46,795 1
94	29,926 7	121	38,519 8	148	47,113 4
95	30,244 9	122	38,838 1	149	47,431 7
96	30,563 2	123	39,156 4	150	47,750 0
97	30,881 5	124	39,474 6	—	—
98	31,199 7	125	39,792 9	—	—
99	31,518 0	126	40,111 2	—	—
100	31,836 2	127	40,429 5	—	—
101	32,154 5	128	40,474 8	—	—
102	32,472 7	129	41,066 0	—	—
103	32,791 0	130	41,384 3	—	—
104	33,109 3	131	41,702 6	—	—

^a This is sometimes referred to as “unit pitch circle diameter”.

Annex B (informative)

Equivalent chain designations

See [Table B.1](#).

Table B.1 — Equivalent chain designations

Chain pitch mm	ISO chain number	Previous ISO chain number
6,35	25	04C
9,525	35	06C
12,7	40	08A
12,7	41	085
15,875	50	10A
19,05	60	12A
25,4	80	16A
31,75	100	20A
38,1	120	24A
44,45	140	28A
50,8	160	32A
57,15	180	36A
63,5	200	40A
76,2	240	48A

Annex C (informative)

Method of calculating chain minimum dynamic strength

C.1 “ANSI” series chain

For 41 chain only:

$$F_d = K_s \times A_i \times p^{(-0,0008p)}$$

For all other chains:

$$F_d = K_s \times 0,118 \times p^{(2 - 0,0008p)}$$

where

F_d is the chain minimum dynamic strength at 3×10^6 cycles, in newtons (N);

A_i is 12,01 mm² for 41 chain only;

K_s is

- 115 N/mm² for 41 chain only;
- 134 N/mm² for chains up to and including 160;
- 139 N/mm² for chains 180 and above;

p is the chain pitch, in millimetres (mm).

NOTE Constant K_s is increased from 134 N/mm² to 139 N/mm² to allow for the reduction in the test specimen length from 5 free pitches to 3 free pitches when conducting the dynamic strength test.