

Hydrodynamic Drives Terminology – SAE J641 FEB83

SAE Recommended Practice
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Report of the Hydrodynamic Drive Committee, approved January 1951, last revised by the Transmission and Drivetrain Technical Committee February 1983.

[Since the torque converter and fluid coupling have become a commonly used component of automatic transmissions in industry, the SAE appointed a committee to standardize terminology, test procedure, data recording, design symbols, and so forth, in this field. The following committee recommendations will facilitate a clear understanding for engineering discussions, comparisons, and the preparation of technical papers.

The recommended usages represent the predominating practice or the acceptable practice. Where agreement is not complete, alternates have been included for clarification. EXAMPLE: two systems of blade angle designations are described. Consequently when a blade angle is specified, the system should be designated.

This SAE Recommended Practice deals only with the physical parts and dimensions and does not attempt to standardize the design considerations, such as the actual fluid flow angle resulting from the physical blade shape.]

Hydrodynamic Drive—As contrasted with electrical or mechanical and so forth, is the type of drive that transmits power solely by dynamic fluid action in a closed recirculating path.

Fluid Coupling—A hydrodynamic drive which transmits power without

ability to change torque. (Torque ratio is unity for all speed ratios.) See Fig. 1.

Torque Converter—A hydrodynamic drive which transmits power with ability to change torque. (Torque ratio is a function of speed ratio.) See Fig. 2.

Element—An element consists of a single row of flow directing blades. See Fig. 4.

Member—A member is an independent component of a hydrodynamic unit such as an impeller, reactor, or turbine. It may comprise one or more elements. See Fig. 4.

Stage (Single-, Two-, Three-, and so forth)—A stage is a turbine element interposed between elements of other members. The number of stages is the number of such elements of the turbine member. See Figs. 2 and 4.

Phase (Single-, Two-, Three-, and so forth)—Applied to a torque converter refers to the number of functional arrangements of the working elements when the functional change is produced by a one-way clutch or other mechanical means such as a clutch or brake. See Figs. 2 and 4.

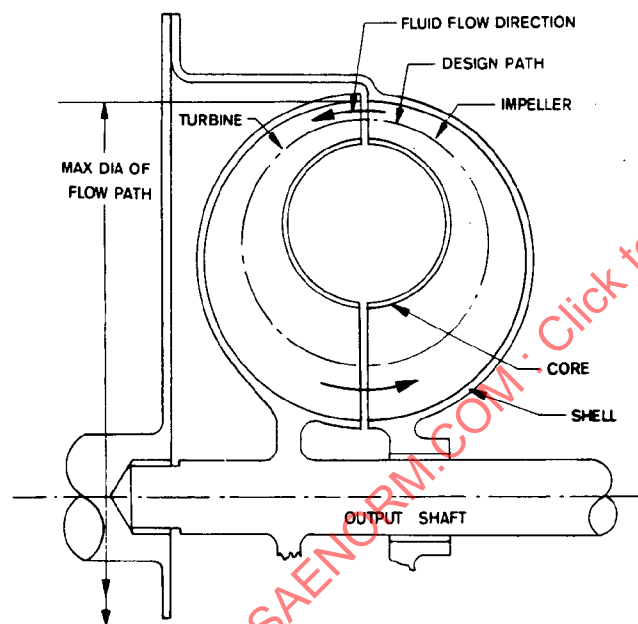


FIG. 1—FLUID COUPLING

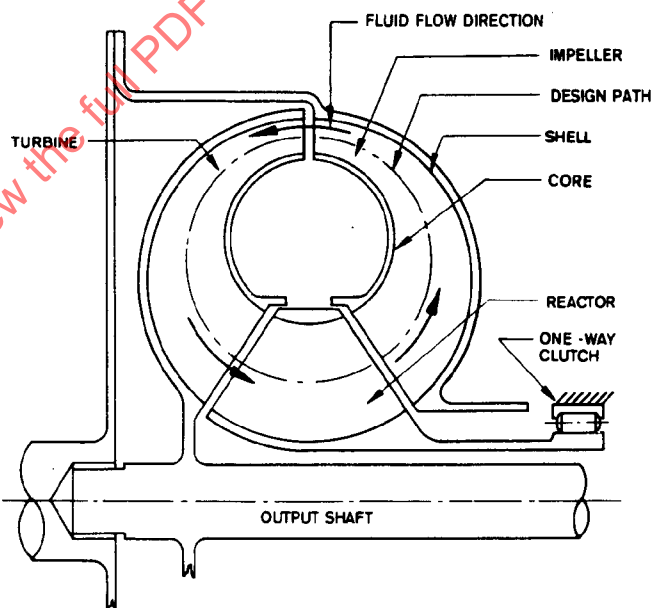
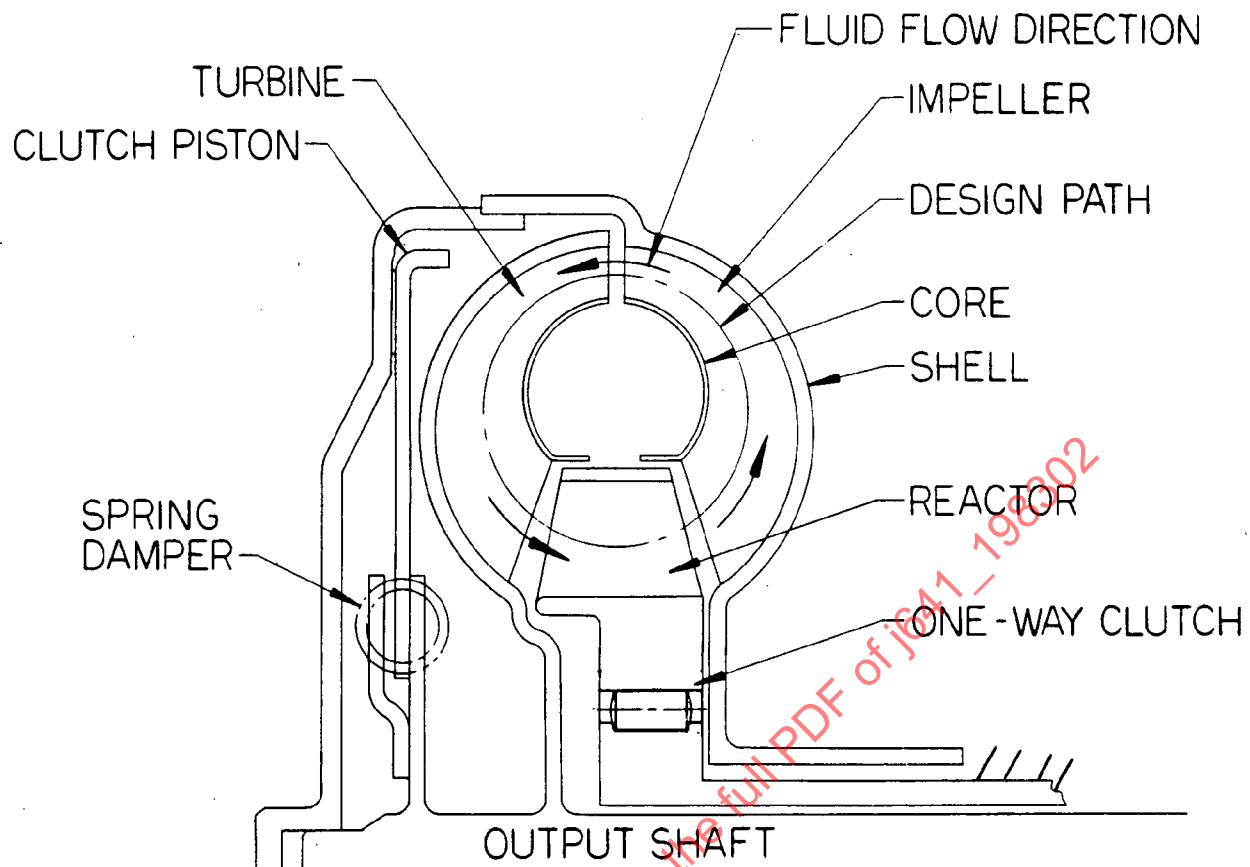
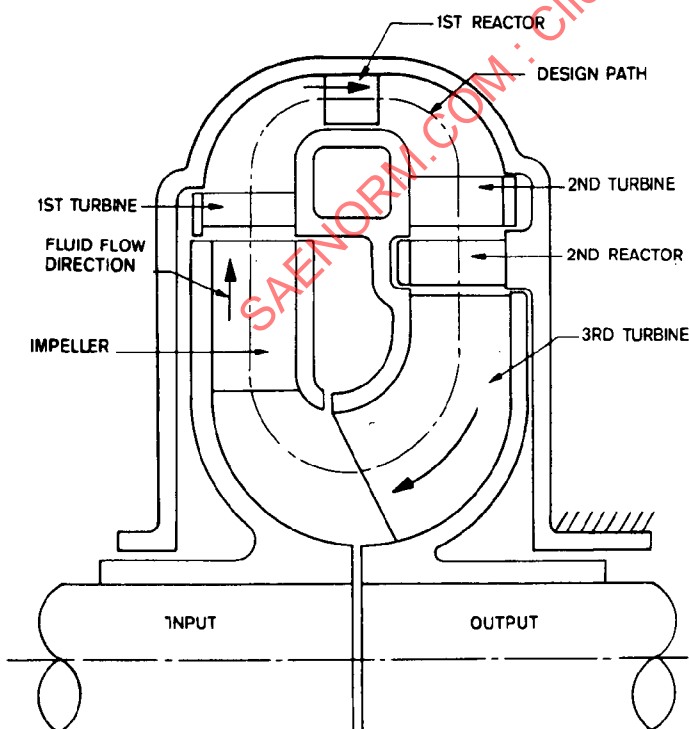


FIG. 2—TWO PHASE, SINGLE STAGE TORQUE CONVERTER
(SINGLE PHASE, SINGLE STAGE IF ONE-WAY CLUTCH IS DELETED)



φ FIG. 3—LOCK-UP TORQUE CONVERTER



φ FIG. 4—THREE MEMBER, SIX ELEMENT, SINGLE PHASE, THREE STAGE TORQUE CONVERTER

Impeller—Designates the power input member.

Turbine—Designates the output member.

Reactor—Designates the reaction member.

One-Way Clutch—See SAE J1087 (May 1974).

Lock-Up Torque Converter—A hydrodynamic torque converter using a φ clutch to provide a direct mechanical drive. See Fig. 3.

Clutch Piston—A friction element device which mechanically couples φ the impeller and turbine. See Fig. 3.

Damper—A device that reduces the amplitude of torsional vibration φ to the output. See Fig. 3.

Nomenclature of Multiple Members—Nomenclature of multiple members of basically the same function in both polyphase and multistage torque converters should be named in the order of fluid circulation in normal operation:

first impeller
second impeller
and so forth

first turbine
second turbine
and so forth

first reactor
second reactor
and so forth

Blade—Within an element, designates the means of directing fluid flow.

Variable Blade—Designates a blade provided with control means to vary the angular position and thus vary the direction of fluid flow.

Torus Section—Designates the confines of a flow circuit in a radial plane of a torque converter or fluid coupling.

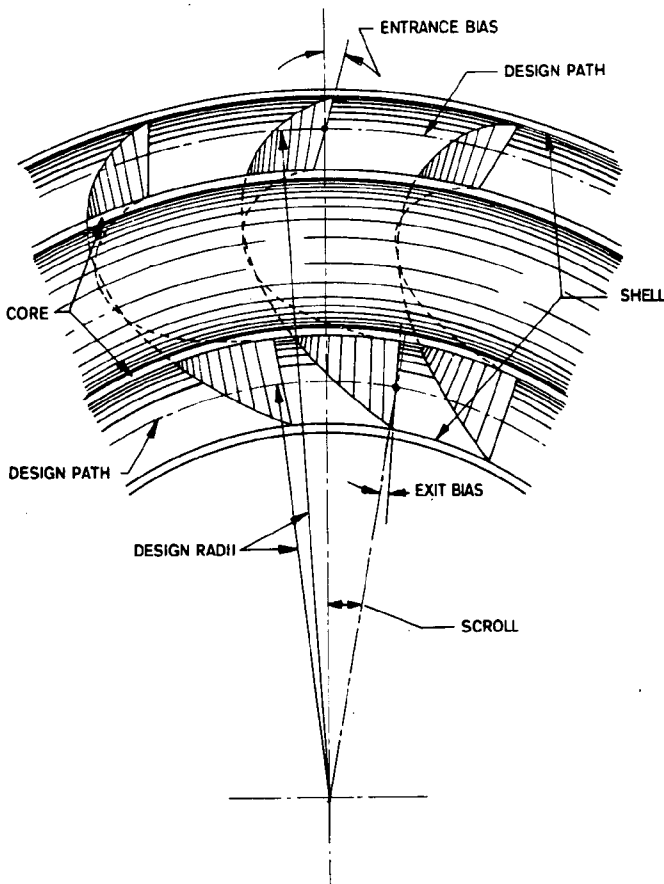
Shell—Designates the outside wall of the torus section in any member. φ See Figs. 1 and 2.

Core—Designates the inside wall of the torus section in any member. φ See Figs. 1 and 2.

Design Path—The path of the assumed mean effective flow and is used for definition of blade angles, entrance and exit radii, and so forth. See φ Figs. 1 and 2.

Bias (Entrance and Exit)—At the entering and exit blade edges, designates the angular variance with respect to an axial plane at the design φ path. The angle is measured as viewed in an axial direction. See Fig. 5.

Scroll—The angle between the two planes containing the intersection



φ FIG. 5—BLADE TERMINOLOGY (TURBINE)

of the design path and the entering and leaving edges of the blade when that blade does not lie in one axial plane. See Fig. 5.

Torque Converter Size—In general terms is designated by the maximum diameter of the flow path. See Fig. 1.

Design Radii (Entrance or Exit)—Design radii of any member are taken at the point of intersection of the design path with the theoretical blade φ edges. See Fig. 5.

Slip—Designates the difference between input and output rpm. It may also be expressed as a percent of input.

Speed Ratio—Designates the output speed divided by the input speed.

Torque Ratio—Designates the output torque divided by the input torque.

Capacity Factor for a member is the rpm divided by the square root of the torque.

$$K = \frac{N}{\sqrt{T}}$$

Stall Torque Ratio—Designates the torque ratio with a stalled turbine.

φ **Stall Start**—Accomplished by restraining the vehicle with the brakes, opening the throttle fully and subsequently releasing the brakes after the engine has attained maximum stall speed.

Stall Speed—Designates the input speed in revolutions per minute with a stalled turbine at a specified input torque.

Racing Speed—Designates the input speed in revolutions per minute with a free turbine at a specified input torque.

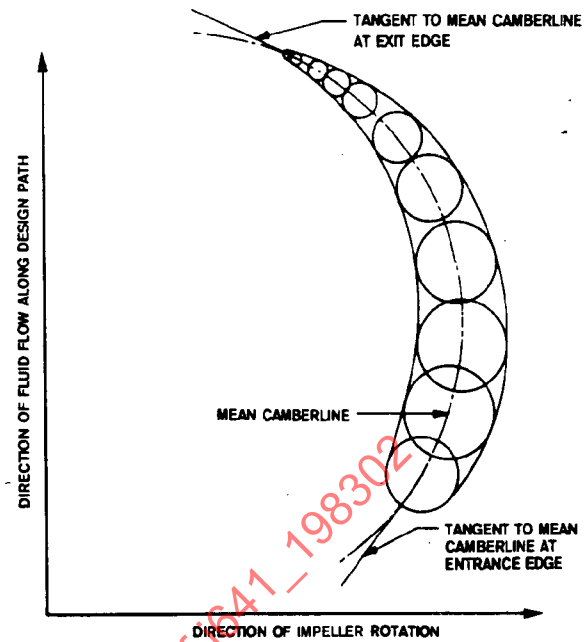
Torque Conversion Range—Designates the range of operation where torque multiplication exists.

Coupling Range—Designates the range of operation at which torque ratio is unity.

Coupling Point—Designates the point where the torque conversion range ends and the coupling range begins.

Hydrodynamic Unit Charge Pressure—Designates the externally applied pressure under which the hydrodynamic unit operates.

Mean Camberline—Mean camberline is the locus of the centers of the series of circles which are tangent to both surfaces of the blade profile, φ see Fig. 6.



φ FIG. 6—DEVELOPED SECTION OF BLADE AT INTERSECTION WITH DESIGN PATH SURFACE

BASIC ORIENTATION OF BLADE PROFILE

DIRECTION OF FLUID FLOW ALONG DESIGN PATH

DIRECTION OF IMPELLER ROTATION

	BLADE ANGLE SYSTEM "A"	BLADE ANGLE SYSTEM "B"
EXIT	65°	25°
REACTOR	-10°	100°
ENTRANCE		
EXIT	60°	150°
TURBINE	45°	
ENTRANCE		45°
EXIT	10°	80°
IMPELLER		
ENTRANCE	-15°	105°

φ FIG. 7—BLADE ANGLE SYSTEMS